HEAT TRANSFER CHARACTERISTICS OF MICRO-DROPLET IMPACTING ON HIGH-TEMPERATURE WALL IN PYROLYSIS OF METHYL RICINOLEATE

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The rapid heating of the material is a key factor in the pyrolysis of methyl ricinoleate, which is mainly affected by the droplet size and velocity of methyl ricinoleate droplet impacting on high-temperature wall. The process of millimeter-droplet of methyl ricinoleate impacting on high-temperature wall at different droplet sizes and velocities were studied by high-speed photography technology and multiphysics simulation. The simulation results were basically consistent with the experimental results. Based on this model, the process of micro-droplet impacting on high-temperature wall was investigated, obtaining the heat rates of droplet at different droplet sizes and velocities. The processes of 38 ~ 58 µm droplet impacting on hightemperature wall at velocity of $10 \sim 30$ m/s were investigated. As the droplet at the initial temperature of 27 °C and droplet size of 38 µm impacting on a high-temperature wall of 480 °C at a speed of 20 m/s, the average heating rate of the droplet reached 10^7 °Cs orders of magnitude. The results showed that faster heating rates were obtained at smaller droplet size and faster droplet velocity. Hence, the rapid heating of the material can be achieved by the micro-droplet impacting on stable high-temperature wall.

Key words: *droplet impinging, phase field, micro-droplet, simulation, methyl ricinoleate*

1. Introduction

Methyl ricinoleate (MR) produced from castor oil can be pyrolyzed at high temperature to form methyl undecylenate (UAME) and heptanal (HEP). UAME is mainly used to produce polyurethane polymers, cosmetics and pharmaceuticals. HEP is a chemical intermediate used in the production of spices and cosmetics [1].

In the traditional MR pyrolysis process, the MR diluted with water is preheated and pumped directly into the high-temperature electric furnace for pyrolysis [2–4]. In this pyrolysis process, heat is transferred mainly by conduction, which inevitably leads to a temperature gradient from the outer wall of the reactor to the central area of the reactor [5]. This increases the deep pyrolysis and coking of the reactor and thus reduces the yields and selectivities of the products.

In order to solve this problem, our group developed the reactors that utilize microwave assisted heating and induction heating coupled with spray heat transfer for MR pyrolysis [6–8]. During the

pyrolysis process, MR is preheated and pumped to the pressure swirl nozzle to break up into microdroplets [9, 10]. Then these droplets hit the high-temperature wall at high speed to transfer heat and pyrolyze. MR droplets can be heated quickly and uniformly [11]. The reaction temperature and heating rate are the key factors in MR pyrolysis [12, 13]. The yields and selectivities of these reactors on the target products are significantly higher than with conventional reactors. Therefore, it is necessary to study the flow and heat transfer properties of MR droplets impacting a high-temperature wall. However, direct simulation of spray pyrolysis is impossible due to the detailed hydrodynamic principles such as droplet ejection, droplet interaction with the heated metal surface, etc. Hence the modeling currently exceeds the computational power. From the report of Abedinejad et al. [14], it is known that when spray-generated droplets evaporate, the collisions between droplets are negligible. Thus, by simulating a single droplet impact without transient consideration of the reaction process, the complexity of the entire spray pyrolysis problem can be reduced and a simplified problem can be simulated numerically [15].

The flow and heat transfer properties of droplets impinging on a high-temperature wall are affected by the droplet size, droplet velocity, fluid properties and roughness of the solid wall, which makes the study of the flow and heat transfer properties of droplets impinging on a high-temperature wall extreme complicated [16–18]. Currently, the flow and heat transfer properties of droplets impacting a high-temperature wall are being studied through experimental research and numerical simulation.

The motion process of droplet impact on a high-temperature wall has often been studied using high-speed photography technology. Li et al. [19] observed the deposition process of high impacting energy droplets on a horizontal surface by high-speed photography technology. The influence of droplet impacting velocity and size on the impacting process was studied in detail. The results showed that the droplet velocity was the main factor affecting the droplet diffusion during the impacting process. Negeed et al. [20] proposed that the droplet impacting velocity, diameter and other parameters will change the contact time of the droplet impacting on the wall. Li et al. [21] studied the flow and heat transfer process of droplets impacting on high-temperature walls by high-speed photography technology and numerical simulation. The results showed that the heat flux of the droplet is affected by the initial velocity, and the heat transfer efficiency increased with the increasing of the droplet velocity. However, the previous studies mainly focused on the process of low-velocity millimeter droplets impacting a high-temperature wall, and a simple similarity comparison between large droplets and microdroplets used in establishing previous models is not accurate enough. In addition, there are few studies on the flow and heat transfer properties of microdroplets impacting the high-temperature wall.

Numerical simulation provides an efficient way to reveal the heat transfer mechanism of droplet impact. Alhashem et al. [22] proposed a model of droplet heat transfer on a vertical plate by combining theory and numerical simulation. For investigate the flow and heat transfer characteristics of micro-droplet impacting on high-temperature wall, level-set method, volume of fluid (VOF) method, Euler-Lagrange method and phase field method were often used to simulate the flow and heat transfer process of droplet impacting [23–25]. In the numerical simulation of the droplet impacting process, it is easier to correctly realize the influence of surface tension and other parameters by using the level set method to track the moving interfaces [26]. The phase-field model used a fixed grid to treat the interface as a thin with limited-size transition layer between different phases [27]. There is no need to

use moving meshes to satisfy the idealized incremental boundary conditions along the interface. In addition, the phase-field approach is highly adaptable and easy to account for thermal coupling effects [28]. However, there are relatively few studies on the simulation of droplet impacting process using a phase field model.

In this study, to simulate the movement and heat transfer process of an MR droplet impacting on a high-temperature wall at various particle sizes and velocities, a phase field model was established in this study. This model couples the fluid flow and heat transfer involved in the single droplet impacting on the high-temperature wall. A high-speed camera was employed to record the flow process of a millimeter-droplet of MR impacting on a high-temperature wall at a high velocity in order to confirm the accuracy of the model. The simulation results for heat flux were compared with the available experimental data. Then, using this model, the flow and heat transmission properties of the MR micro-droplet impacting on the high-temperature wall were examined, and the heating rates of droplet were determined. This work may be useful for refining the atomization procedure in spray pyrolysis.

2. Experimental

2.1. Experimental set up

The experimental setup was shown in Fig. 1, including a tank, a peristaltic pump (BTJ100A, Baoding Guangzhi constant flow pump manufacturing company), a needle (24G, Jiangxi Hongda Medical Devices Group Co., Ltd.), a heating device with glass surface with the roughness is less than 0.05 μ m (DB-XWJ, Shanghai Bangxi Lichen Instrument Technology Company), a 100W LED lamp as light source and a high-speed camera (Photron, Fastacm SA-X2, Japan) matching a micro lens (AF micro-Nikkor ED 200mm f/4 D IF). The peristaltic pump is connected to the needle through a silicone tube, the inner diameter of needle is 0.3 mm, and the high-speed camera is connected to the computer. The high-speed camera lens is parallel to the surface of the heated plate. The temperature error of the top microcrystalline panel (200 mm × 300 mm) can be controlled at ± 1 °C.



Fig. 1. Diagram of experimental device.

2.2. Experimental procedure

The MR used in the experiment was purified by distillation, and the purity was 98 % determined by gas chromatography. First, the glass surface of the heating device was heated to the set temperature (480 °C). MR was sent to the needle through a peristaltic pump at flow velocity of 6 mL/h to form a single droplet. Then the droplet was impacted on high-temperature wall. The initial temperature of the droplet is room temperature. The impacting velocity of the droplet can be controlled by the distance between the tip and the heated microcrystalline plate, and t = 0 s was specified as the initial contact time of the droplet on the high-temperature wall. The high-speed camera is used to record the impact process of the droplet. In this study, the shooting speed of the high-speed camera is set to 5000 frames per second, which is convenient for quantitative analysis, the pixel of the recorded image is 1024×1024 . The time interval between two neighboring images is 0.2 ms, the height difference of the droplet on different images and the corresponding time interval can be recorded to calculate the droplet wall speed, the measurement accuracy is ± 0.05 m/s. The droplet size and spreading diameter were determined according to the ratio scale. The spreading coefficient β was introduced to analyze the spreading degree of droplet, which was expressed as:

β

$$= l/\mathbf{D}_0 \tag{1}$$

l——spreading diameter of droplet, µm;

 D_0 —initial diameter of droplet, μ m.

The static contact angle between the MR droplet and the Glass surface was measured to be about 35° using the OCA-20 data physics instrument according to the static fixation method as shown in Fig. 2. During the experiment, the droplets fall by gravity only. Each set of experiments was repeated three times.



Fig. 2. Contact angles of MR on the high temperature wall.

3. Numerical calculation

3.1. Geometric model of droplet impacting on high-temperature wall

In the process of MR droplets impacting on high-temperature wall, a series of behaviors, such as

droplets impingement heat transfer occurrence with high-temperature wall, are extremely random and complex. In this paper, the numerical model was built by using the COMSOL Multiphysics software. The simulation was based on the two-dimensional axisymmetric model established by the experimental results taken by the high-speed camera. The contact angles of MR on high temperature wall inputted in the simulation model were 35 °. The geometric model was shown in Fig. 3, composed of droplets, hightemperature wall and air. The specific values of the initial velocity are obtained experimentally. At the initial moment, the lower end of the MR droplet is in contact with the top of the high-temperature wall. As shown in Fig. 3(a), the square represented the air and the semicircle represented the droplet. Fig. 3(b) was three-dimensional model by the rotary transform of Fig. 3(a). In the related research carried out in this paper, the phase-field coupled heat transfer method will be



Fig. 3. The numerical simulation model of single droplet impacting on hightemperature wall, (a) two-dimensional axisymmetric model; (b) threedimensional model.

used to numerically simulate the behavior of MR droplets impinging the high-temperature wall, and the heat flux density in the droplet is analyzed. In order to save the simulation cost and speed up the calculation speed, the geometric model of the droplet impacting on high-temperature wall was simplified:

(1) The influence of high-temperature wall thermal radiation was ignored;

(2) The temperature of high-temperature wall remained constant;

(3) The mass remained constant and the reaction process is not considered during the droplet impacting on the high-temperature wall;

(4) The shape of the droplet at the initial time was regular spherical;

(5) Gas phase air was an ideal gas and the initial air velocity is 0 m/s;

(6) The fluid was incompressible.

3.2. Governing equation

The governing equation of the two-phase flow is a conservation equation based on mass, momentum and energy, which are shown as follows:

$$\rho\left(\frac{\partial \vec{u}}{\partial t}\right) + \rho \nabla \cdot (\vec{u}\vec{u}) = -\nabla P + \nabla \cdot \mu [\nabla \vec{u} + (\nabla \vec{u})^T] + \rho \vec{g} - \overrightarrow{F_{st}},\tag{2}$$

$$\nabla \cdot \vec{u} = 0, \tag{3}$$

$$\rho \frac{\partial T}{\partial t} + \rho \nabla \cdot (\vec{u}T) = \nabla \cdot \left(\frac{k}{c_p} \nabla T\right) + Q_e, \tag{4}$$

where, \vec{u} is velocity vector, ρ is density, P is pressure, μ is dynamic viscosity, T is temperature, \vec{g} is gravity vector, $\vec{F_{st}}$ is surface tension per unit volume at the two-phase interface, c_p is specific heat capacity, k is thermal conductivity, and Q_e is source term.

Phase field method is a common method to study interface phenomenon in multiphase flow. The phase interface between the two phases can be expressed by the phase field parameter Φ , which can be described by the Cahn-Hilliard method [29]:

$$\partial \phi / \partial t + \vec{u} \cdot \nabla \phi = \nabla \gamma \cdot \nabla \mathbf{G},\tag{5}$$

where, γ is migration parameters controlling diffusion scale, m³·s/kg; *G* is chemical potential at the phase interface, Pa.

The formulas for the expressions γ and G are as follows:

$$\gamma = X e_{pf}^2, \tag{6}$$

$$G = \lambda \left[-\nabla^2 \phi + \left[\phi(\phi^2 - 1) \right] exp(2) \right], \tag{7}$$

where, X is slip, m·s/kg; e_{pf} is interface liquid film thickness, m; λ is energy mixing density, N. The density and dynamic viscosity of the two-phase flow are defined as follows:

$$\rho = \rho_1 V_{f1} + \rho_2 V_{f2} \tag{8}$$

$$\mu = \mu_1 V_{f1} + \mu_2 V_{f2} \tag{9}$$

The subscripts 1 and 2 represent different phases in the formula. The volume fractions Vf in two phases are as follows:

$$V_{f1} = (1 - \phi)/2 \tag{10}$$

$$V_{f2} = (1+\phi)/2 \tag{11}$$

 $\phi = \pm 1$ denotes two-phase fluid. $\phi = 0$ represents the phase interface.

The surface tension per unit volume F_{st} at the interface of the two phases can be obtained by the gradient of the phase field variable multiplying the chemical potential of the system:

$$\overline{F_{st}} = G\nabla\phi \tag{12}$$

3.3. Simulation procedure

In previous studies, the Sauter mean diameter of MR after atomization through a pressure swirl nozzle was between $20 \sim 70 \ \mu\text{m}$, and the velocity was about $10 \sim 30 \ \text{m/s}$. Hence, the process of 38, 48 and 58 μm MR droplets impacting on a 480 °C wall at a speed of 10, 20 and 30 m/s in 50 μs were simulated by the model. The t = 0 s was specified as the initial contact time of the droplet impacting on high-temperature wall. The spreading coefficient β of the droplet in 50 μ s was obtained and the average temperature of the droplet was counted by simulating the spreading length of the droplet. Then the heat flux of the high-temperature wall was analyzed to investigate the heat transfer process of the droplet impacting on the high-temperature wall. For analyzing heat transferring features of on droplet impacting high-temperature wall, an accurae estimate of wall heat transfer coefficient is requied. The localized heat flux q and the heat flow Q at wall were calculated using Fourier's law:

$$q = -k_f \left(\frac{\partial T}{\partial n}\right) = h \left(T_w - T_f\right)$$
(13)

$$Q = qA_f \tag{14}$$

The localized heat transfer conefficient of wall is expressed as follows:

$$h = -\frac{k_f}{(T_w - T_f)} \left(\frac{\partial T}{\partial n}\right) \tag{15}$$

where *h* is wall heat transfer coefficient, A_f is contact area between fluid and solid, T_W is temperature of high-temperature wall, T_f is fluid temperature, k_f is fluid thermal conductivity, and *n* is coordinate normal to wall.

3.4. Boundary conditions

The upper and right sides of the computational domain were set as pressure boundary conditions in the laminar flow module. The outlet relative pressure is 0 MPa. The bottom side of the computational domain was set to the boundary of the high-temperature wall at 480 °C.

The fluid-structure interface follows the basic conservation equations, satisfying the conditions of thermal flux and temperature conservation at the interface of fluid and wall:

$$\begin{cases} q_f = q_w \\ T_f = T_w \end{cases}$$
(16)

3.5. Mesh generation

The geometric mesh of the model was divided by the method of physical field control. The model selected the free tetrahedral mesh element. The mesh size was 100 μ m × 70 μ m. The number of coarsening, conventional, refinement, finer and ultra-fine meshes were 2150, 3402, 5177, 23164 and 85768, respectively. As shown in Fig. 4, the mesh size had little effect on simulation results of heat flux at 50 μ s. Considering the calculation cost and accuracy, the model used the finer mesh, and the number of grids was 23164.



Fig. 4. Grid independence verification.

4. Results and discussion

4.1. Simulation model verification

4.1.1 Simulation of droplet spreading characteristics and experimental verification

Using high-speed photography equipment, the millimeter-scale MR droplet impacting process on the high-temperature wall was examined and compared with the outcomes of the simulation. The results as shown in Fig. 5. It is evident from the image processing software analysis that the experiment's needle-produced droplets had a diameter of roughly 2 mm. The initial droplet temperature is 23.5 °C. The experimental and simulated droplet spreading diameters achieved their maximums at 4 ms, 7.17 mm and 7.08 mm, respectively, when the droplet struck the wall at a speed of 0.58 m/s. The spreading coefficients in the simulation and the experiment were 3.12 and 3.15, respectively. There was a strong correlation between the experimental and simulated spreading diameters.



Fig. 5. (a) experimental process; (b) simulation process of MR droplet impacting on high-temperature wall.

4.1.2 Simulation of heat flux distribution and experimental verification

The millimeter-scale simulation solutions of heat flux and contact temperature at the wall are compared with the available experimental data in order to validate the simulation model. Fig. 6 shows the comparison of heat flux variation which was calculated through this model with the experimental date from Moon et al. [30] under the condition of a water droplet impacting on the copper wall. The



Fig. 6. Comparison between simulated and actual heat flux variation.

initial droplet diameter is 2 mm, initial impact velocity is 0.5783 m/s, initial droplet temperature is 23.5 °C. It is found that the variation trend of heat flux and the influence trend of temperature predicted by the simulation results are consistent with the experimental data. The deviation between the simulation results and experimental data is tiny at lower initial wall temperature. Therefore, the model can be used to simulate the flow and heat transfer process of droplets impacting on high-temperature wall.

4.2. Impact process of micro-droplet on high-temperature wall

4.2.1 Effect of droplet size on impact process

The effect of the droplet size on spreading coefficient was presented in Fig. 7 (a). The microdroplet can reach the maximum spreading diameter in 10 μ s, which was 400 times faster than the millimeter-sized droplets. The variation trend of the spreading coefficient was in good agreement with the experimental results of Visser et al. [31]. As the velocity of the droplet was 20 m/s, the spreading coefficient increased with the increasing of droplet size, but the smaller droplet can reach the maximum spreading diameter faster. In addition, the droplets spreading coefficient curve remained basically parallel in the retraction stage, which indicated that the retraction velocity of the droplet was less affected by the droplet size.





4.2.2 Effect of droplet velocity on impact process

As shown in Fig. 8, the maximum spreading coefficient of droplet increased with the increasing of droplet velocity. The results was the same as the simulation study of Zhang et al. [32]. The effect of the droplet velocity on spreading coefficient was presented in Fig. 7 (b) when the droplet diameter was 58 μ m. The maximum spreading coefficient of the droplet increased with the increasing of the droplet velocity. The time required to reach the maximum spreading coefficient decreased with the increasing of the velocity. Droplet of 58 μ m reached the maximum spreading diameter in 5 μ s at the velocity of 30 m/s, which was half of the time than the droplet of 10 m/s. The spreading coefficient of droplet remained parallel in the retraction stage at different velocity, which indicated that the retraction velocity of the droplet was less affected by the droplet velocity in the droplet impacting on high-temperature wall.



Fig. 8. Spreading of droplet impacting on high-temperature wall at different droplet velocities.

4.3. Heat transfer process of micro-droplet on high-temperature wall

4.3.1 Heat flux of the wall and temperature distribution of the droplet

The temperature distribution of 58 μ m droplet impacting on high-temperature wall at a speed of 10 m/s and the average temperature of droplet increased by 117 °C in 1 μ s were shown in Fig. 9. The high-temperature section of the droplet was impacted to the area which was far from the center of the wall by the low-temperature section of the droplet in the process of droplet spreading, and the low-temperature section. Therefore, the heating rate of the droplet increased with the increasing of the spreading diameter and the decreasing of the time to reach the maximum spreading diameter. During the period of reaching the maximum spreading diameter (0 ~ 16 μ s), the average heating rates of droplet was 1.8 × 10⁷ °C/s. The average heating rate was obviously higher than the study of Yu et al. [11], which was 5.0 × 10⁴ °C/s. This deviation may be caused by the flow process of droplets impingement have been considered in this study. Overall, the spray MR micro-droplet could be flash heated.



Fig. 9. Temperature distribution (58 μm, 10 m/s) of a single droplet impacting on hightemperature wall.

The radial distribution of high-temperature wall heat fluxes at different times were shown in Fig. 10. The heat flux was significantly higher than millimeterdroplet impingement [30]. The temperature difference between the high-temperature wall and the droplet, and the heat flux decreased with the increasing of the heat transfer time. In addition, the heat flux gradually decreased as the horizontal distance increased. The reason was that the temperature difference between the edge of the droplet and the high temperature wall is small.



4.3.2 Effect of droplet velocity on the heat transfer of the droplet

The heat flux distribution of the droplet impacting on the high-temperature wall at the speed of 10, 20, 30 m/s were shown in Fig. 11. In Fig. 11 (a), the differences of heat flux at the radial distance of 20 μ m to 40 μ m in the process of initial impacting were due to the difference in droplet spreading diameter which was caused by different velocity. In addition, droplet will also adhere to the high-temperature wall in the initial stage of micro-sized droplet impacting on high-temperature wall. The

small temperature difference between the wall and the attachment section of droplet resulted in a low heat flux area. As shown in Fig. 11 (b), the heat flux decreased with decreasing droplet velocity at the time of 50 μ s. The mass and heat transfer processes of the droplet were slow at a low droplet velocity.



Fig. 11. Effect of droplet velocity on the heat flux distributions of high-temperature wall at (a) 1 μ s; (b) 50 μ s.

4.3.3 Effect of droplet size on the heat transfer of the droplet

The effect of droplet size on the distribution of wall heat flux was shown in Fig. 12 at the velocity of 20 m/s. As shown in Fig. 12 (a), owing to the diffusion diameter of the droplet increased with the increasing of the droplet size, the range of wall heat flux increased with the increasing of droplet size at 1 μ s. The heat flux in the central area of the high-temperature wall was unchanged. This was because the central area of the high-temperature wall was still contacted with the low-temperature area of the droplet at 1 μ s.

As shown in Fig. 12 (b), the heat flux in the central area of the high-temperature wall decreased with the decreasing of droplet size at 50 μ s. The temperature difference between the smaller droplet and the high-temperature wall was smaller. The reason was that the smaller droplet had a longer contact time with the high-temperature wall after reaching the maximum spreading diameter.



Fig. 12. Effect of droplet size on the heat flux distributions of high-temperature wall at (a) 1 µs; (b) 50 µs.

5. Conclusions

In this paper, a phase field model was established to simulate the process of a single droplet impacting on high-temperature wall. The process of rapid heating of MR micro-droplet was visualized, and the droplet impacting process was recorded and analyzed to verify the accuracy of the model by high-speed photography technology. The simulation solutions of heat flux were compared with the existing experimental data. Based on this model, the process of micro-droplet impacting on high-

temperature wall was investigated at different droplet sizes and velocities. The main conclusions were as follows.

(1) The increase of droplet size will lead to the increase of spreading diameter and the time to reach the maximum spreading diameter. The droplet spreading diameter increased with the increasing of droplet velocity, and the time to reach the maximum spreading diameter was short. The droplet of 58 μ m reached the maximum spreading diameter in 5 μ s at the velocity of 30 m/s, which was half of the time at the velocity of 10 m/s. The droplet size and velocity had little effect on the droplet retraction speed.

(2) The droplet with higher velocity can reach the larger maximum spreading diameter in the lower time, and the droplet with smaller size can reach the maximum spreading diameter in the lower time. At the maximum spreading diameter of the droplet, the heating rate increased with the decreasing of droplet size and the increasing of droplet velocity.

The heat transfer rate of the droplet impacting on high-temperature wall was on the order of 10^7 °C/s. This showed that the rapid heating of the material can be achieved by the micro-droplet impacting on the stable high-temperature wall. Thus, this study could provide reference to the optimization of the atomization process in MR pyrolysis.

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