PARAMETRIC ASSESSMENT OF A SOLID OXIDE FUEL CELL-ORGANIC RANKINE CYCLE HYBRID SYSTEM FOR CLEAN POWER GENERATION

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This paper investigates the energy performance of a hybrid system utilizing waste heat from a solid oxide fuel cell to drive an organic Rankine cycle. The study evaluates the organic Rankine cycle performance with various fluids, considering their global warming and ozone depletion potentials. To enhance system efficiency, the organic Rankine cycle is integrated with the solid oxide fuel cell to utilize its waste heat. A comprehensive mathematical model is developed to simulate the coupled system, encompassing all components. This study focuses on the parametric analysis of the organic Rankine cycle, considering crucial working parameters such as operating fluids, evaporator temperature, and pressure ratio. Cogeneration of the solid oxide fuel cell fuel by feeding the organic Rankine cycle increases system performance by 20%. Notably, the R290 fluid emerges as the most efficient among the proposed working fluids. These findings offer valuable insights into hybrid system energy performance, emphasizing the potential for efficiency improvement through waste heat utilization.

Key words: Clean Power Generation, Waste Heat Recovery, Solid Oxide Fuel Cells, Organic Rankine Cycle, Working Fluids.

1. Introduction

In recent decades, researchers and scientists have shown a strong interest in renewable energy as a sustainable solution to the world's ever-increasing energy needs. This shift in focus represents a collective effort to redefine our energy consumption patterns in order to reduce the environmental impacts associated with traditional reliance on fossil fuels, particularly the significant CO2 emissions they generate [1-5]. As such, the main driver of research efforts is the pursuit of discovering and implementing alternative, cleaner sources of energy production that can effectively contribute to reducing climate change and our carbon footprint. The Organic Rankine Cycle (ORC) is one of the technologies attracting attention in the search for sustainable energy solutions. This innovative cycle uses the principle of waste heat recovery, making it applicable to a wide range of industries. By recovering heat that would otherwise go unused for examples the geothermal heat [6-8], solar thermal energy [9-12], waste heat from industrial [13] and waste heat from the fuel cells [14-16], the ORC offers not only a path to greater energy efficiency, but also opportunities for cost reduction and environmental stewardship. Its adaptability and scalability make it a promising candidate in the transition to a greener, more sustainable energy landscape [17-19]. Although the Organic Rankine Cycle (ORC) has established itself as a viable technology for recovering low-grade waste heat, its relatively modest thermal efficiency is a notable limitation to its technical applications. One of the main challenges in improving ORC thermal efficiency is the variance between the temperature-enthalpy profile of the waste heat sources and the organic working fluid used in the ORC system [20, 21]. Addressing this gap is essential to overcoming this obstacle. Additionally, researchers have focused on analyzing the energy efficiencies of ORC modules. Although, researchers have focused on analyzing the energetic efficiencies of ORCs modules. Krzysztof Matuszny et al. their study focuses on a combined system consisting of a solid oxide fuel cell (SOFC), a hot water storage tank (HWST), and an absorption refrigeration (AR) cycle. Using modelling software, the thermodynamic properties are evaluated, and a sensitivity analysis is performed. Optimal conditions result in a coefficient of performance (COP) and CCHP efficiency of 0.806 and 85.2% for the LiBr-H2O system and 0.649 and 83.6% for the NH3-H2O system, respectively. In addition, under optimum conditions, the SOFC achieves a net electrical efficiency of 57.5% and a net power output of 123.66 kW [22]. Moreover, Can Liu et al. This study combines an ORC system with a PEMFC cooling water recirculation system to increase energy efficiency through heat recovery. Comparing R134a and R245fa, R134a is more efficient (11.33 % vs. 9.16 %). Increasing stack temperature improves efficiency, while increasing current density decreases efficiency. Energy efficiency trends are in line with energy efficiency trends [16]. And Kumar et al. have introduced a new approach to improve the energy production of a combined SOFC, GT, and ORC system. Their research focused on the use of intercooling heat to increase the energy and exergy efficiency of the system. Their results showed that incorporating an intercooler improved both energy efficiency and exergy performance [23]. In addition, Ragini et al. focused on improving energy production processes beyond traditional fossil fuels. Their study focused on a hybrid cycle integrating gas turbines, organic Rankine cycles, and solid oxide fuel cells (SOFCs). Using an ORC with three different working fluids - R141b, R245fa and R236fa - the system utilizes waste heat from the SOFC-GT system. Simulation results showed an increase in efficiency of between 8% and 12%, with R236fa offering superior performance and efficiency [24]. And Rabah et al. This study examines the energy performance of an Organic Rankine Cycle (ORC) using three organic fluids: Toluene, R245fa, and R123, and examines how operating temperatures and organic fluid selection affect cycle efficiency. Toluene was found to be the most efficient fluid with an energy efficiency of 7.45%. In addition, the study examines the effect of evaporating temperature on cycle efficiency and shows a significant increase with higher temperatures, most notably for toluene, whose efficiency increases from 10.5 % to 21.45 % between 80 °C and 150 °C. However, those studies did not consider the environmental impact of the selected working fluids [25].

Based on the mentioned literature, the aim of this study is to perform a comprehensive parametric and energetic study of an Organic Rankine Cycle (ORC) driven by waste heat generated by a Solid Oxide Fuel Cell (SOFC). In particular, the study introduces a novel approach by carefully selecting working fluids according to their ozone depletion potential

(ODP) and global warming potential (GWP). The selected fluids include R290, R152a, R600 and R245Fa. In addition, research is focused on optimizing overall system efficiency by integrating the SOFC and efficiently recovering its waste heat. This multi-faceted approach is expected to provide valuable information on how to improve energy conversion efficiency while considering environmental sustainability factors.

2. Numerical model

The SOFC is a versatile technology that can be powered by a variety of fuels, making it a promising option for clean energy production. In this study the operation parameters of the solid oxide fuel cell (SOFC) remain constant.

Fig. 1 shows the basic diagram of a system combining an SOFC fuel cell and an ORC Organic Rankine Cycle. Each component is shown to reflect the arrangement of solid oxide fuel cells, turbine, condenser, pump, heat recovery boilers, compressors, and heat exchangers. Ambient air is compressed from point 1 to point 2, then passes through a heat exchanger to be heated at point 3 before entering the solid plate fuel cell cathode. Methane gas is also compressed from point 4 to point 5, then passed through a heat exchanger and heated at point 6. Similarly, water is pumped from point 8 to point 9, heated at point 10 using the same technology, and then mixed with fuel at point 7 to produce hydrogen gas, which feeds the anode. After mixing with the fuel circulating from point 6 to point 7, the mixture reacts to produce hydrogen gas, which is used to power the anode. The gas emitted by the anode and cathode, at points 11 and 12, respectively, is then sent to a thermal transformer to recover heat from the evaporator, which is used to supply a power circuit to operate this cycle. The operating parameters of the SOFC and the ORC are showed in Table 1 [26].

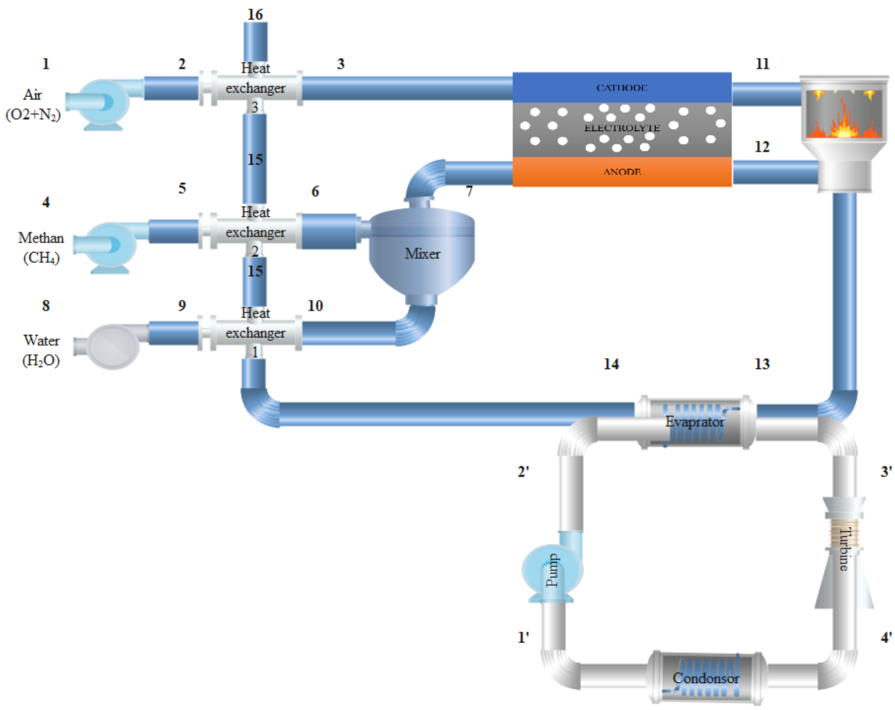

Figure 1. Schematic design for the suggested system.

Table 1: System Operating Conditions.

Parameter	Value	Unit
R_{cont}	0	Ω.m²
L_{an}	5×10^{-4}	m
L_{ca}	5×10^{-5}	m
L_e	1×10^{-5}	m
L_{int}	3×10^{-3}	m
A_{cell}	0.01	m²
N_{FC}	11000	-
R_{stc}	2.5	-
F	96485	-
U_f	0.75	-
U_a	0.15	-
J	8000	A/m²
T_{Pc}	873	K
P_{ref}	101.15	kPas
Pressure drops in the stack	2	%
Pressure drops in the exchanger	3	%
τ_{Ca}	1.19	-
τ_{Cc}	1.19	-
LHV_{CH4}	802361	J/mol
ORC		
Evaporator temperature	90	°C
Isentropic pump efficiency	0.9	-
Isentropic turbine efficiency	0.85	-
Low pressure	100	kPa
High pressure	1000	kPa

The thermophysical properties of the suggested organic fluids used as working fluids in the organic Rankine cycle are listed in Table 2 [27].

Table 2: Thermophysical Properties of the Four Organic Fluids

Working fluids	R1270	R-600	R-290	R-245Fa
Chemical formula	C_3H_6	C_4H_{10}	C_3H_8	$C_3H_3F_5$
Molar mass in (kg/mol)	42.08	58.1	44.10	134.05
Critical temperature in (°C)	92	152	370	154
Critical pressure in (bar)	46.6	36.709	42.5	36.5
ODP	0	0	0	0
GWP	0	3	<10	950

Several assumptions were made to model the system [28,29]:

- The composition of the air is assumed to be 21 % oxygen and 79 % nitrogen.
- All substances are treated as ideal gases.

- The system operates in a steady state.
- The solid oxide fuel cell (SOFC) is assumed to be adiabatic with respect to the environment.
- The methane reforming reaction is assumed to be complete, so there is no methane at the mixer outlet.
- The SOFC inlet and outlet are assumed to have the same temperature and pressure.
- The temperature difference between the inlet and outlet is kept constant.
- Turbines, compressors, and pumps are assumed to be adiabatic with isentropic efficiencies.
- The evaporator and condenser outlet flows are assumed to be saturated.

2.1. System modelling

To simulate the solid oxide fuel cell's performance, an electrochemical model incorporating the Butler-Volmer equation, Fick's model, and Ohm's law was employed. This model accounted for activation, concentration, and ohmic losses, respectively. The generated energy of the SOFC is written as:

$$P_{out} = N_{cell} \times V_{SOFC} \times j \times A_{cell} \tag{1}$$

where N_{cell} number of cells, A_{cell} area of cell, V_{SOFC} SOFC overall voltage.

The considered voltages are shown in Table 3. [32-34].

$$V_{sofc} = E_{nernst} - V_{act} - V_{conc} - V_{ohmic} \tag{2}$$

Table 3. Voltage calculation

Voltage	Equation
Nernst reversible voltage	$E_{nernst} = E_0 + \frac{RT}{2F} ln(\frac{P_{H_2} P_{O_2}^{0.5}}{P_{H_2O}})$
Activation losses	$V_{act} = V_{act,an} + V_{act,ca}$
	$V_{act,an} = \frac{RT}{2F} sinh^{-1}(\frac{j}{2 \times 0.65 \times 10^4})$
	$V_{act,ca} = \frac{RT}{2F} sinh^{-1}(\frac{j}{2 \times 0.25 \times 10^4})$
Ohmic losses	$V_{ohm} = (R_{contact} + \rho_{an}.L_{an} + \rho_{ca}.L_{ca} + \rho_e.L_e + \rho_{int}.L_{int}).j$
Concentration losses	$V_{act} = V_{conc,an} + V_{conc,ca}$
	$V_{conc,an} = -\frac{RT}{2F}[ln(1 - \frac{j}{j_{as}}) + ln(1 + \frac{P_{H_2}.j}{P_{H_2O}.j_{an}})]$
	$V_{conc,ca} = -\frac{RT}{2F} ln(1 - \frac{j}{j_{ca}})$

2.2. Thermodynamic model

Thermodynamic modelling is performed, and mass balances are defined as follows [35]:

$$\sum \dot{m}_{in} = \sum \dot{m}_{out} \tag{3}$$

The 1st law of thermodynamics is used to formulate the energy balance of any system and is written in general terms as shown below:

$$\sum \dot{m}_{in} h_{in} + \sum \dot{Q}_{in} + \sum \dot{W}_{in} = \sum \dot{m}_{out} h_{out} + \sum \dot{Q}_{out} + \sum \dot{W}_{out} \quad (4)$$

Using the ORC-SOFC system and the principles of the 1st law of thermodynamics as a foundation, Table 4. shows the system performance parameters.

Table 4. System performance parameters

Term	Equation
SOFC AC power	$\dot{W}_{SOFC,AC} = P_{out} \eta_{inv}$
Air compressor	$\dot{W}_{AC} = \dot{m}(h_2 - h_1)$
Fuel compressor	$\dot{W}_{FC} = \dot{m}(h_5 - h_4)$
Water pump	$\dot{W}_{wp} = \dot{m}(h_9 - h_8)$
Net electrical power	$\dot{W}_{SOFC,net} = \dot{W}_{SOFC,AC} - (\dot{W}_{AC} + \dot{W}_{FC} + \dot{W}_{wp})$
The SOFC energy efficiency	$\eta_{FC} = \frac{\dot{W}_{SOFC}}{\dot{m}_{CH_4} LHV_{CH_4}} \times 100$
ORC power	$\dot{W}_{ORC} = \dot{m}(h_4 - h_3)$
ORC Net power	$\dot{W}_{ORC,net} = W_{ORC} - W_{FP}$
HRSG input energy	$\dot{Q}_{in,HRSG} = \dot{m}_{14} h_{14} - \dot{m}_{3'} h_{3'}$
The ORC energy efficiency	$\eta_{ORC} = \frac{W_{ORC,net}}{\dot{Q}_{in,HRSG}}$
System efficiency	$\eta_{FC} = \frac{\dot{W}_{SOFC} + \dot{W}_{ORC,net}}{\dot{m}_{CH_4}.LHV_{CH_4}}$

2.3. Validation

To validate the simulated SOFC configuration, a comparative analysis was performed with the theoretical study of Bafekr et al. [36] and the experimental investigation of Tao et al. [37], A comprehensive review of the relevant literature was conducted to validate the SOFC model [36, 37]. The polarization validation is shown graphically in Fig. 2, which illustrates the variation of voltage versus current density.

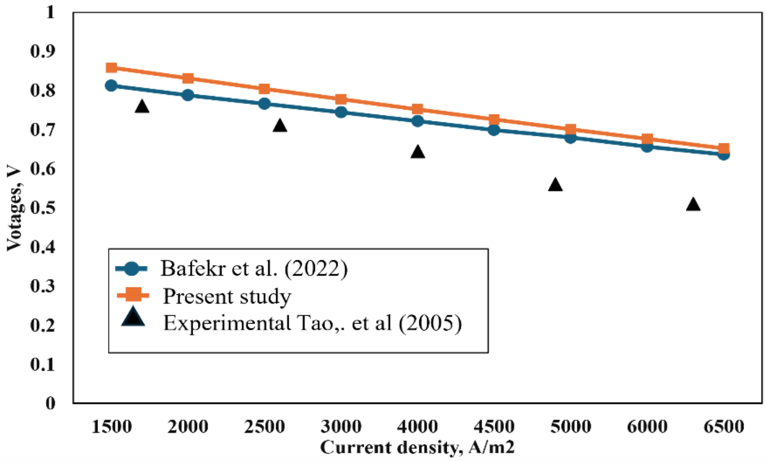

Figure 2. Validation of the proposed SOFC model.

3. Results and discussion

3.1. Effect of evaporator temperature

This section examines the effect of evaporator temperature on ORC and overall system performance. In addition, all proposed working fluids for the ORC are examined to assess their effect on system performance. Fig. 3 illustrates the effect of evaporator temperature on ORC system performance with four organic fluids. For each organic fluid, the effect of evaporator temperature was studied by varying it from 50 °C to 110 °C. The results show that increasing the evaporator temperature significantly increases the power produced by the organic Rankine cycle. According to the data, the R290 fluid stands out as the most efficient, generating an optimum power of 150 W at 50 °C, which increases proportionally to reach its maximum value of 190 W at 90 °C. Next, the R152a fluid performed in the same evaporation temperature range, with an initial power of 130 W that increases proportionally to about 180 W. Results for the R600 fluid follow, with power ranging from 85 W to 110 W in the same evaporation temperature range, while those for the R245Fa fluid, with lower power, are between 65 W and 90 W. moreover the Fig.3 shows the evolution of the power produced by the combined ORC-SOFC system as a function of the evaporator temperature. The results show a simultaneous increase in power as the evaporator temperature increases. The R290 fluid in the organic Rankine cycle stands out as the best performer, with an initial power of 720 kW at 50 °C, gradually increasing to 750 kW at 90 °C. Similarly, the use of R152a fluid in the same temperature range gives very good results, with an initial power of 700 kW and a proportional increase to about 740 kW. Next, the results for the R600 fluid show a power range from 660 to 720 kW in the same temperature range, followed by the R245Fa fluid which gives a lower power in the same temperature range, from 630 to 699 kW.

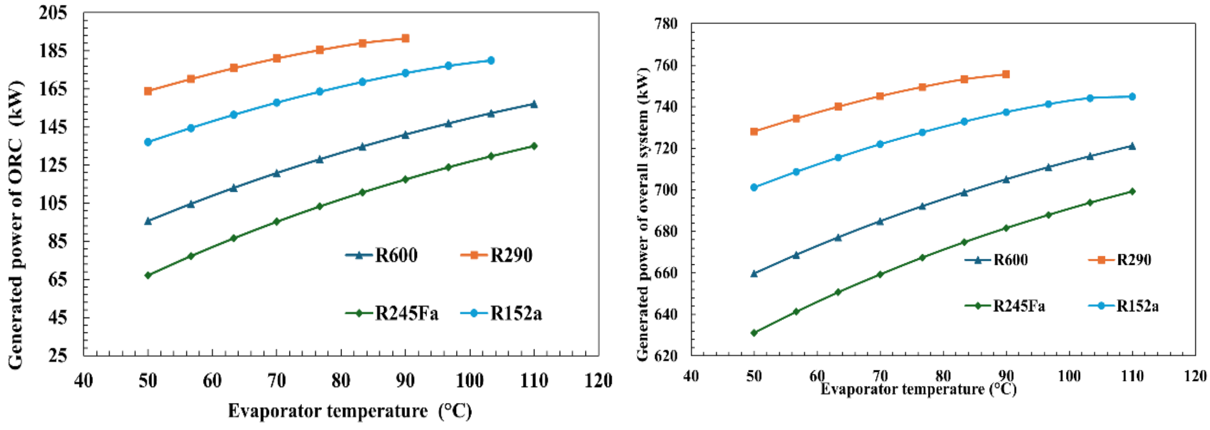

Figure 3. Effect of the evaporator temperature on the ORC power output and the overall system.

Fig. 4 illustrates the effect of evaporator temperature on the efficiency of the ORC cycle. The efficiency of the Organic Rankine Cycle increases proportionally with increasing evaporator temperature, ranging from 0.23 to 0.26 as the temperature increases from 50 °C to 90 °C when R290 fluid is used in the cycle. When R152a is used, the cycle efficiency increases proportionally with increasing temperature from 0.19 to 0.25. When R600 fluid is used, the cycle efficiency varies between 0.13 and 0.22 over the same temperature range, while with R245Fa the efficiency varies between 0.09 and 0.19. In addition to fig. 4 shows the evaporator temperature affects the efficiency of the combined system. In fact, an increase in evaporator temperature results in an improvement in system efficiency. For the organic fluid R290, the system efficiency increases from 0.57 to 0.59 as the evaporator temperature increases from 50°C to 90°C. For R152a, the cycle efficiency increases proportionally with temperature, from 0.55 to 0.58. Results for R600 show efficiencies ranging from 0.52 to 0.56 over the same temperature range, while for R245Fa efficiencies range from 0.49 to 0.55 over the same temperature range.

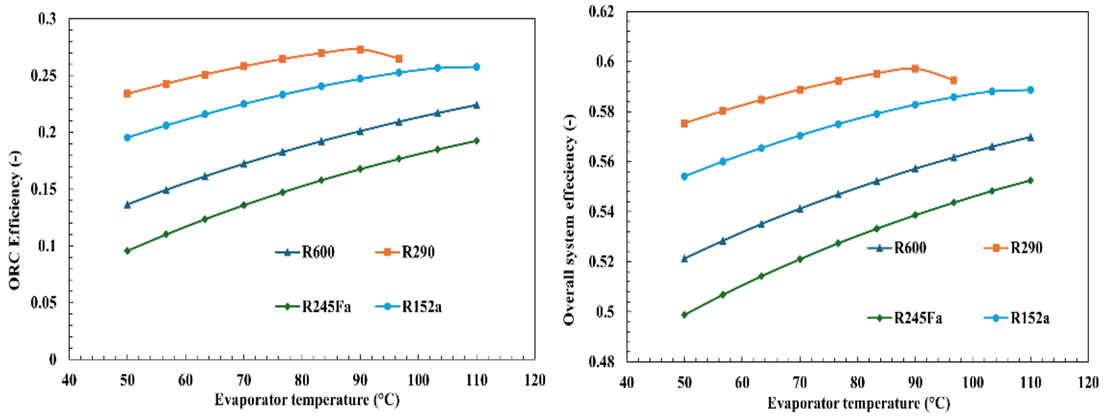

Figure 4. Effect of evaporator temperature on ORC and overall system efficiency.

3.2. Effect of compression ratio

According to Fig. 5, the effect of increasing the compression ratio on the power produced by the organic Rankine cycle varies according to the organic fluids used. The R290 fluid produces the highest power, ranging from 188 to 192 kW, while the results obtained with the R152a fluid are satisfactory, with a power ranging from 170 to 174 kW. The results obtained with R600 fluid are around 141 kW, followed by those obtained with R245Fa fluid, which reach around 118 kW. In all cases, a slight reduction in the power generated by the cycle is observed as the compression ratio increases. As shown in Fig. 5, the effect of increasing the compression ratio on the power output of the combined SOFC/ORC system varies with the organic fluids used in the Rankine cycle. The R290 fluid achieves the highest power output, ranging from 752 kW to 756 kW. The results obtained with R152a are satisfactory, ranging from 734 kW to 730 kW. The results obtained with R600 fluid are around 706 kW, followed by those obtained with R245Fa fluid, which reach around 683 kW.

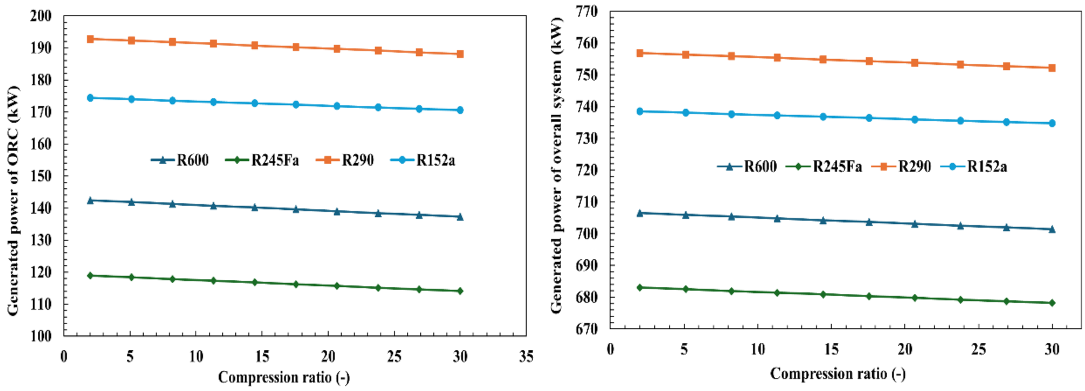

Figure 5. Effect of compression ratio on the generated power of the ORC and overall system.

Fig.6 shows the effect of increasing the compression ratio on the efficiency of the ORC cycle using different organic fluids in the cycle. The cycle efficiency peaks when R290 fluid is used in the Rankine cycle at 0.27. This is followed by R152a at 0.24, R600 at 0.2, and R245Fa at 0.15. Moreover fig. 6 illustrates the effect of increasing the compression ratio on the efficiency of the combined SOFC/ORC system using different organic fluids in the organic Rankine cycle. It can be seen that the use of R290 fluid in the Rankine cycle leads to a maximum efficiency of 0.59, followed by the use of R152a fluid with an efficiency of 0.58, then the use of R600 fluid with an efficiency of 0.55, and finally the use of R245fa fluid with an efficiency of 0.53.

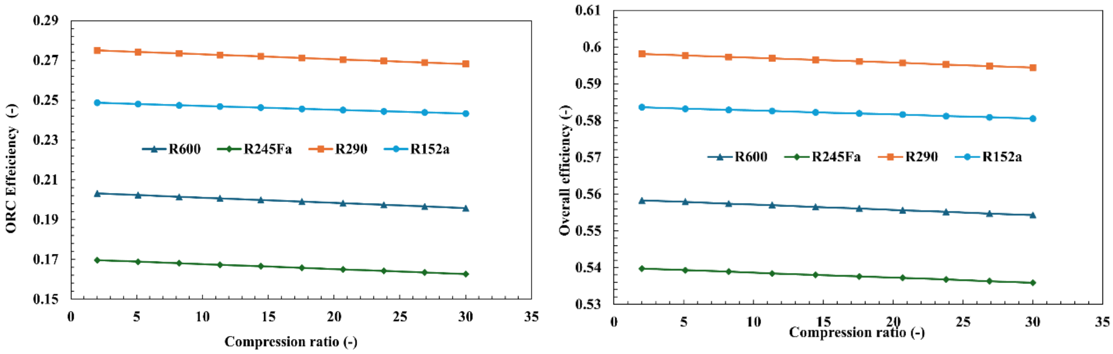

Figure 6. Effect of compression ratio on the ORC and overall system efficiency.

4. Conclusion

This study examines the results of a cogeneration system integrating an SOFC and an ORC. Several parameters were evaluated, including evaporator temperature and ORC compression ratio, as well as several environmental fluids. The results of the study are shown below:

- The results show that increasing the evaporator temperature increases the performance of the organic Rankine cycle, with variations depending on the organic fluids used. The R290 fluid stands out as having the best performance, followed by the R152a, R600 and R245Fa fluids.
- The study also showed that evaporator temperature affects the efficiency of the combined system, with an increase in evaporator temperature leading to an improvement in overall system efficiency. It is also noteworthy that all four organic fluids show a slight decrease in cycle power and overall efficiency as the compression ratio increases.
- These results highlight the importance of organic fluid selection and compression ratio in the design and optimization of integrated ORC systems with SOFCs. Finally, these results have significant implications for energy applications, providing valuable information on the potential performance of different organic fluids in integrated ORC/SOFC systems, as well as the benefits of compression ratio optimization.

Nomenclature

$\mathbf{A}_{\text{cell}}$ – Activation cell area [$\boldsymbol{m^2}$]
$\mathbf{E}_{\text{nernst}}$ – Nernst Reversible Voltage [V]
j – current density [A/$\boldsymbol{m^2}$]
F – Faraday number [C/mol]
K – Equilibrium constant [-]
L – Thickness of SOFC layer [m]
LHV – lower heating value [J/mol]
$\mathbf{N}_{\text{cells}}$ – Number of the fuel cell stack [-]
$\dot{\mathbf{n}}$ – Mole rate [mol/s]
$\boldsymbol{P_i}$ – Partial pressure [bar]
$\mathbf{P}_{\text{out}}$ – Generated Power of the Fuel cell [kW]
$\dot{\boldsymbol{Q}}$ – Heat rate [kW]

$\boldsymbol{R}$ – Ideal gas constant [J/mol.K]
T – Temperature [K]
V – Voltage [V]
$\dot{\boldsymbol{W}}$ – Work rate [kW]
Greek symbols
$\boldsymbol{\eta}$ – Efficiency [%]
$\boldsymbol{\rho}$ – ohmic resistance [ohm]
Subscripts
an – Anode
ca – cathode

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