## SURVEY OF AIR BATTERY THERMAL MANAGEMENT ON THE AUTONOMOUS MOBILE ROBOTS

## by

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With the development of computer technologies and battery systems, mobile robot systems have gained an important place in our lives. The Li-ion batteries, which attract attention for portable applications due to their high power output, light weight, and no memory effect, must operate in a limited temperature range in order to operate safely and for a long application life. This case makes it necessary to use battery thermal control systems that will provide temperature control for Li-ion battery applications. Within the scope of our study, Li-ion battery thermal management systems were investigated in detail. A detailed literature review was carried out on air thermal management systems, with their application and systemic convenience. Due to restricted packaging space for battery in mobile robots, cooling will be more challenging in the future. The air thermal management system, which has limited thermal management capability due to the low thermal properties of the air, is a thermal management method that will be sufficient especially for low scale applications.

Key words: Li-ion battery, battery cooling, air temperature control, control air battery thermal management system, air cooling

#### Introduction

Autonomous mobile robots that reduce manpower in the industry and increase system efficiency have attracted significant attention from the production industry, with the development of computer and software technologies and the Industry 4.0 industrial revolution [1, 2]. Autonomous mobile robots have become tools used in many operations of the production industry today, including production-lines and storage processes [3-5]. Each component of robot systems consist of many mechanical and electrical systems has an ideal operating temperature [6]. Thermal management for each of these components is very important for both itself and for their interactions with each other [7].

The temperature control of Li-ion battery power generation systems, which is widely preferred in mobile robot systems, is of vital importance for battery systems to operate safely and efficiently in the required cycle life [8]. The electrochemical reaction during the battery charge-discharge process creates reaction heat and joules heat due to ohmic resistances during ion transfer, and this state causes the battery temperature to increase exponentially during operation if cooling is not performed during battery operation [9]. It is recommended that battery

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systems operate in the 15-35 °C operating temperature range, with the maximum temperature difference not exceeding 5 °C. In case the battery operated outside these temperature ranges, irreversible damage may occur to the battery. For this reason, various active or passive thermal management systems are used to to keep battery temperature under control.

Battery heat generation processes are difficult to understand, due to the difficulty of knowing how the electric current is distributed in the battery and variation of battery electrochemical reaction rate with time and temperature [10]. In order to carry out the cooling system design and optimization studies in a rational way, a good understanding of the battery heat generation processes is required. For this reason, many studies have been carried out on the electrochemical modelling of battery heat generation processes depending on the battery charge discharge rate and battery operating temperature [9-11]. Wu [9] studied temperature distribution and heat transfer models for simulate the temperature distribution. Kwon and Park [12] carried out numerical simulations to measure the amount for battery degradation and thermal runaway of battery operating wide temperature range. Tang et al. [13] noted that the heat generation on the battery surface is uneven, that the temperature is not homogeneously distributed on the battery surface and a higher amount of heat build-up in areas close to the cell current collection ports. It is not possible to achieve battery temperature distribution homogeneity and this case creates an important constraint for thermal management. It was concluded that the battery heat generation and heat generation inhomogeneity increased as the discharge rate increased, and the ambient temperature had little effect on the average heat generation rate.

Mobile robot systems consist battery power generation systems and with it associated of many electronic components and devices. Battery thermal control is not only for the safe and efficient operation of the battery, it is also necessary for the safety and efficient operation of affected by battery temperature systems and devices working in connection with the battery [4-7].

Air and liquid thermal management systems are predominantly preferred methods for battery temperature control. The air thermal management system, which is the subject of this study, a simple, low weight, leak free air thermal management system that uses air-flow to remove battery heat is the oldest preferred thermal management method for controlling battery temperature. It is not suitable to be used alone in extreme conditions and high power batteries, because of the low heat transfer ability of the air and low thermal management ability.

In our study, a detailed literature summary about air thermal management system, one of the basic battery thermal management systems, was prepared. It was observed that the studies focused on design optimization and design proposal. Due to it would be quite expensive and complex for studies to proceed only through experimental studies, it was seen that the studies were mainly carried out as numerical analysis and experimental verification.

## Battery thermal management systems

Operation of battery systems outside the lower and upper operating temperature limits seriously battery capacity and service life deterioration, and even in consequence of thermal leaks such as combustion and explosion, it creates irreversible safety problems [14-18]. Thermal management systems are required to heat or cool the battery depending on the operating and ambient conditions of the battery system for the battery to meet the required performance and service life [17, 19].

#### Air thermal management system

It is a thermal management method of controlling the temperature of the battery by natural air convection or forced air convection by the fan [19]. Although natural-convection

usually does not provide adequate thermal control, it may be adequate for thermal management at low discharge rates. Zhang *et al.* [20] studied the thermal control by natural air convection of a battery pack consisting of nine cells placed in various lay-outs. From straight row, square and circular placed lay-out battery packs, the straight row were obtained lower battery temperature in natural air convection, and then with square and cylindrical lay-out, respectively. Then, the effect of the distance between the batteries on the thermal properties of the battery pack was examined, it was seen that increasing the distance between the battery cells decreased the maximum temperature and the maximum temperature. However, it should not be forgotten that improves the thermal properties with increasing the distance between the battery cells will increase the battery pack volume. Yu *et al.* [21] observed the necessity of using a forced air battery thermal management system for battery pack safety at higher discharge rates, although the system thermal properties can be kept within the desired range at discharge rates as low as 0.5C in natural air cooling for the battery scheme in which they work.

Considering the studies with natural air convection and their results, it can be concluded that the forced air cooled air thermal management system provides a more reliable thermal control than the natural air convection thermal management system. In air thermal management system studies, researchers mainly focused on forced air convection air thermal management and focused on flow system modelling and flow system optimization [19, 22]. In flow system modelling and optimization studies various focal points were focused on. These, the cell arrangement in the battery pack, the cooling air inlet and outlet positions are design initiatives that will change the flow characteristics.

In so far as the flow path and characteristics of the batteries change depending on the arrangement of the cells in the battery pack, depending on the cell lay-outs, the thermal management capability of the battery pack varies. In the study of Fan *et al.* [23] examined the effects on thermal management efficiency of 32 cylindrical battery cells are arranged in a staggered and cross layouds of cells aligned in the battery pack, see fig. 1. The battery pack with the aligned cell arrangement was found to provide the best thermal performance. Following the aligned cell lay-out the most efficient cell arrays are staggered, diagonal, respectively. Aligned battery pack arrangement for thermal management required up to by 2% lower fan power consumption than cross arrangement. The most important factor for thermal management is the number of cells in the module, beyond cell arrangement and flow rate.





Since the cooling air-flowing in the battery pack absorbs the heat of the battery cells, the ability of the air to absorb heat decreases, battery temperatures increase in the direction of cooling air-flowing, higher temperatures occur in the battery pack output cells. To prevent this situation, by making design modifications to the air inlet section in order to improve the cooling efficiency and temperature homogeneity of the battery pack, Shahid and Agelin-Chaab [24] de-

signed a system, where the cell temperatures are higher that will provide fresh air supply to the outlet cells. With this design initiative, which provides fresh air supply to the outlet cells, 18.3% lower maximum temperature and 54.6% lower temperature difference were achieved. Na *et al.* [25] designed a battery pack that provides reverse air-flow between layers, in which the air-flow area is bisected by a polyamine plate for lower the temperature of the battery output cells and to increase the temperature uniformity of the battery pack. Due to the thermal interactions of the reverse air-flow layers, lower maximum temperature and higher temperature uniformity were achieved compared to the single-layer system. Similar to Na *et al.* [25] and Xu *et al.* [26] used a design that divides the air-flow field into layers. Unlike the other study, the flow direction was the same in the air-flow layers. At 2 m/s inlet air velocity, 3C discharge velocity, the single-layer design provided 5.15% less max temperature and 16.01% less maximum temperature difference than the unlayered design.

In battery packs cooled by one-way air-flow, another method used for to prevent of increase the output cells to higher temperatures and the decrease in the temperature homogeneity of the battery pack, has been the design of a system that allows the air-flow to flow back and forth in a certain period with valves that open and close at certain intervals. Mahamud and Park [27] designed and suggested a system that will provide forward-backward air-flow in a certain period, the schematic representation of which is given in fig. 2. It was stated that the shorter the forward-backward air-flow time, the lower the temperature difference and the maximum temperature. The maximum temperature difference and the maximum temperature decreased by 10 °C and 1.5 °C, respectively, compared to unidirectional air-flow with a back-and-forth air-flow period of 120 seconds. Wang and Ma [28] examined the strategies of unidirectional cooling, reciprocating cooling with constant period reciprocating air-flow, and active-controlled reciprocating cooling in their study. With reciprocating air cooling, the battery pack temperature uniformity was significantly improved. With the active control, the cooling power consumption for required cooling has been significantly reduced.



Figure 2. Reciprocating air cooling that provides forward-backward air-flow [27]; (a) flow direction from right to left and (b) flow direction from left to right

Lu *et al.* [29], who did not use the conventional battery module lay-out, created a battery module by being fixed on plates with ventilation holes in six rows of 252 cells, and the way of providing battery temperature control with axial air-flow was followed. In the analyzes carried out, investigated the effects on thermal management of duct size and number of ducts to allow cooling air to flow between battery packs. It was stated that increasing the channel size would not be an effective way to increase the thermal management efficiency, and increasing

the number of channels was more effective. In another study using a battery module with a similar battery module lay-out as Yang *et al.* [30] studied the cooling of battery cells by axial airflow, evaluating the effects of inter-cell radial spacing and cooling air-flow rate. Increasing the radial spacing of the battery pack slightly increases the battery pack average temperature while improving temperature uniformity. In addition, increasing the radial spacing of the cells from 2-10 mm reduced the cooling system parasitic power losses by 95.5%. Increasing the cooling system air-flow rate reduces the battery pack's maximum temperature and minimum temperature, while also improving the temperature uniformity within the battery pack, as it reduces the maximum temperature nore than the minimum. Yang *et al.* [31] propose placing radiators with bionic surface structure to the battery pack consisting of cylindrical cells cooled by axial air-flow. It has been reported that the increase in bionic surface thickness and height effectively reduces the maximum temperature and maximum temperature difference, and the bionic surface shape affects the maximum temperature and maximum temperature difference at a negligible level.

Cooling air inlet and outlet location has been another research topic due to it affects the air-flow path and characteristics, thermal characteristic and thermal management efficiency of the battery module [32-35]. Li *et al.* [34] compared thermal kontrol effect of the fan positions. In terms of thermal performance, the fan on the back of the battery is more advantageous than on the side of the battery was reported. The reason for this is that in the design where the fan is positioned on the side surface of the battery, the heat transfer efficiency and thermal homogeneity are low, since the cooling air continues to heat up inside the battery pack. It has been reported that increasing the fan position and number to increase the effective heat transfer area will increase the thermal management capability. Battery holders have been ignored in studies examining the effects of fan position on thermal management.

Controlling the battery temperature with cooling air-flowing in the tangential direction has been one investigation subject in air thermal management systems. The tangential air-flow is sent to the battery pack cells placed at regular intervals, the cooling air is distributed tangentially to the intercellular channels and leaves the battery pack at varying positions depending on the tangential blower type. Depending on whether the air-flow directions are different or in the same direction, tangential air blower thermal management systems are called *u*-type and *z*-type tangential air blowers, respectively.

In the study of Li et al. [35] optimized the mass-flow rate and intercellular flow channel size for the U-type tangential air-cooled thermal management system. When the inter-cell flow channels of the battery pack are too wide, almost all of the flow will flow from the front flow channels of the battery pack, and the cooling air-flow will not flow to the back flow channels, this case causes a higher maximum temperature in the back cells and a higher temperature difference in the battery pack. Xie et al. [36] evaluated the cooling air inlet-outlet nozzle angles and made optimizations in U-type tangential air blower thermal management. It was determined that the optimum inlet and outlet nozzle angle was  $2.5^{\circ}$ , and the maximum temperature and temperature difference were reduced by 12.82% and 29.72%, respectively. Gocmen and Cetkin [37] used the elevated battery position improve the thermal management performance of the thermal management system in the Z-type tangential blower system operating at fast and ultra-fast discharge rates. With the elevated battery positions, the cooling air showed uniform flow resistance and velocity. Thanks to this design, the maximum temperature between cells has been reduced from 12-0.3 K. A fin was added to the thermal management design to increase the uniformity of intercellular temperature, electrical resistance, and aging rate above 6C discharge rates.

udies in the literature	Battery $T_{max}$ $\Delta T$ $T_{env}$ load $[\circ C]$ $[\circ C]$ $[\circ C]$	1-3C     55.4     3.2     35     The best thermal management performance is the line, square and cylindrical type lay-out, respectively, inside the battery lay-out	1-5C     52.2     6.7     23     In 3C, 4C and 5C discharge rates, the battery pack maximum temperature exceeded 40°C safety upper limit	It has been observed that forced air cooling is necessary for the battery to operate in safe thermal conditions at discharge rates above 0.5C discharge rate. Increasing the air supply flow rate in forced air cooling gradually reduces the maximum temperature	3C     33,50     -     25     Peration. For forced air cooling, the cell arrangement with axial symmetry is very advantageous for cooling ca-pability and thermal homogeneity	It was observed that as the distance between the cells increased, the air-flow rate inside the battery decreased, decreasing flow rate increased the battery temperatures	2C     33,5     1,5     25     The aligned battery lay-out is a better choice for thermal management performance	2C The battery pack with the aligned battery arrangement was found to provide the best thermal performance	2C, 3C31.251.0125It was determined that the 4x4 aligned cell arrangement was the most effective in terms of cooling performance	As the cell numbers in the module increases, the cooling efficiency of the module decreases, while the temperature uniformity increases. In case the row of cells in the module exceeds eight, battery lay-out has no effect on thermal management performance	28     0.11     22     With high discharge rates, the battery temperature and the maximum temperature difference between the cells increase significantly.
ture	$\Delta T$ [°C]	3.2	6.7	7	I		1,5		1.01		0.11
he litera	$T_{ m max}$ $[\circ  m C]$	55.4	52.2	30.5	33,50		33,5		31.25		28
lies in th	Battery load	1-3C	1-5C	0.5-1C	3C		2C	2C	2C, 3C		
nent system stud	The way of work	Experimental	Numerical (ANSYS), experimental	Experimental	Numerical (ANSYS), experimental	Experimental	Numerical (COM- SOL), exp.	Experimental	ANSYS, experimental	Numerical	Numerical
ir thermal managem	Battery type and capacity	2,5 Ah 9 18650 cylindrical cells	20 Ah, pocket type LiFePO <sub>4</sub> cell	22 pcs 2.7 Ah Panasonic NCR18650PF	1,5 Ah Sanyo 18650 lithium-ion cell	6 Ah capacity cell		3,6 Ah, 32 18650 cells	16 serial lithium-ion cells	18650 lithium-ion cell	3,6 Ah battery
ole 1. Ai	Refer- ences	[20]	[38]	[21]	[39]	[22]	[40]	[23]	[41]	[42]	[43]
Tat			7	ŝ	4	9	7	8	6	10	11

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Working output	With the design changes made to the cooling air inlet plenum, 18.3% lower maximum temperature, 54.6% low-er temperature difference were obtained	The design in which vortex generator and jet inlets are added to the inlet plenum, reduced the maximum temperature difference and maximum temperature by 21.5% and 5%, respectively	When the input plenum was placed onp of the battery pack, the maximum temperature was reduced by 9%, the temperature uniformity was improved by about 39%	With the layered reverse air-flow system, lower maximum temperature and higher temperature uniformity were achieved compared to the single-layer system	At 2 m/s inlet air velocity, 3C discharge rate, the single layer design provided 5.15% less max temperature and 16.01% less max temperature difference than the design without layer.	The back-forth air-flow significantly improved the temperature uniformity inside the battery pack	Piston cooling with the back-forth air-flow significantly improved battery pack temperature uniformity compared to unidirectional air-flow cooling	The battery module temperature difference has decreased significantly (from 4.1-1 °C) with active-controlled back and forth air-flow thermal management system design	Forth air-flow is more effective in terms of thermal management in the system where the battery is cooled by back-forth air-flow of the battery	In order to ensure the safe operation of the battery in harsh operating conditions, direct evaporative cooling system, which reduces the air temperature, is integrated into the air inlets of the back and forth air-flow system
$T_{ m env}$ [°C]	23	23			25	35		20	25	
$\Delta T$ [°C]	5	5.69- 7.45		0.0	5	4		1	9	
$T_{ m max}$ [°C]		30-31.7	34.40	37,8	40			<40	20	
Battery load				3C	3C					
The way of work	Numerical (ANSYS)	Numerical (ANSYS), experimental	Numerical (ANSYS), experimental	ANSYS, experimental	Numerical (ANSYS), experimental	Numerical (ANSYS)	Experimental	Experimental	Experimental	Experimental
Battery type and capacity	2.5 Ah, 32 Samsung INR18650-25R	Battery pack consisting of 32 18650	32 pcs 2.5 Ah 18650 lithium-ion cells	300 W 20 pcs cylindrical cell	2.5 Ah, 20 pcs cells	3.6 Ah cylindrical cell	20 Ah 18 LiFePO4 cell	8 26650 cylindrical cells	37 Ah, pocket type cell	9 pcs 3,35 Ah Pana- sonic NCR18650B
Refer- ences	[24]	[44]	[45]	[25]	[26]	[27]	[28]	[46]	[47]	[48]
	12	13	14	15	16	17	18	19	20	21

**Table 1. Continuation** 

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Working output	The effects of air inlet and outlet locations and cooling channel length on thermal management performance were analyzed	Increasing the radial spacing between cells improved the battery pack average temperature and temperature uniformity	Active cooling is required for safe operation of the battery at high discharge rates and ambient temperatures	Increasing the air hole number and hole diameter increase thermal management performance at the expense of very little power consumption. Increasing the inlet air pressure lowers battery module temperatures at the expense of power consumption	It has been reported that the increase in bionic surface fin thickness and height effectively reduces the maximum temperature and maximum temperature difference, the bionic surface shape affects thermal manegement at a negligible level	Having the air inlets and outlets on different sides of the module side surfaces provides better performance	The lowest maximum temperature and the best temperature uniformity were achieved by designing the air inlet and outlet on opposite module side surfaces	As the ventilation hole number is increased, the cooling capability increases depending on the changing air-flow rate, flow path and temperature	The tangential air blower design with the same direction air inlet and outlet provided better thermal management performance	The large flow channels between the battery pack cells cause higher max temperature and temperature difference	The homogeneous distribution of the cooling air inside the battery pack can be improved by the inlet and outlet plenum angle optimizing, the maximum temperature and temperature difference can be reduced
$T_{ m env}$ [ $^{\circ} m C$ ]		25	25-35	25		25		25			
$\Delta T$ [°C]		4>		<5,5	<5.32		$\overline{\vee}$	4.6		6.4	3.1
$T_{\max}$ [°C]		<40		<37	<34	<40	30	37	<35		51
Battery load			1-5C	3-5C	3C	0, 5C 1C		3C	1C		3-6C
The way of work	Numerical (ANSYS 14.5)	Numerical (ANSYS 14.5)	Numerical	Numerical (FLUENT), experimental	Numerical (FLUENT 18.2)	Numerical, experimental	Numerical, experimental	Numerical (ANSYS)		Numerical (ANSYS 17.0)	Numerical
Battery type and capacity	2.2 Ah 18650 lithium-ion cells	32 pcs 26650 LiFePO <sub>4</sub> cell	10 Ah 38120 cylindrical cell	2 Ah, 18.650 cylindrical lith- ium-ion cell	2.6 Ah capacity cylindrical 18650 cells	15,6 Ah 60 pcs 18650 cells	12 LiC battery module	25 pcs 18650 lithium-ion cells	LiFePO <sub>4</sub> prismatic cell	36 pcs cells	2.2 Ah LiFePO4 battery
Refer- ences	[49]	[30]	[50]	[51]	[31]	[32]		[52]	[53]	[35]	[54]
	23	42	25	26	27	28	29	30	31	32	33

**Table 1. Continuation** 

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Working output	As the battery pack air-flow channel size and air intake angle increase, the battery temperature decreases	Cooling channel geometry, channel number and cooling plate material affect the thermal management capability	As the cooling air-flow rate is increased, the heat transfer capability of the system increases, but the pressure losses increase	Increased the flow channel size around the high temperature cells and reducing the maximum temperature difference by more than 29%.	With the distance optimization between the cells, the maximum temperature difference was reduced by more than 60%	Thermal manegement capacity increase, with the optimization of the cooling air inlet and outlet nozzle angle and the distance between cells within the battery pack.	It was found that the maximum temperature and temperature difference of the battery module decreases as the channel size increases	With regionally different cell distance, battery pack volume, maximu temperature and maximum temperature difference have been reduced	The research results showed that the addition of a flow deflector to the air diffuser nozzle and flow channels effectively improves thermal management performance	The thermal performance of the straight flow distributor was better than the arc and parabolic flow distributor	By adding a $60^{\circ}$ angle flow distributor and giving a 5 mm rounding diameter to the edges of battery packet profile to Z-type tangential air blow air thermal management system, the maximum temperature is reducing $36.66\%$
$T_{env}$ [°C]	25			27	27	25				25	
$\Delta T$ [°C]	2.3	1.6	2.6	<2.9	<5			4.1	0.33	~11	
$T_{ m max}$ [ $^{ m CC}$ ]	37.58		39.2	<42.6		<36.5			47.46	~37	
Battery load	1C, 1.5C			4C, 5C	5C						1-5C
The way of work	Numerical (ANSYS)	Numerical (ANSYS)	ANSYS, experimental	Numerical	Numerical (FLUENT)	Numerical, experimental	Numerical (ANSYS 12.0)	Numerical	Numerical (ANSYS)	Numerical (ANSYS 17.0)	Numerical
Battery type and capacity	100Ah		16 Ah battery consisting	2.2 Ah 2 × 36 LiPePO <sub>4</sub> cells	2 × 12 LiPePO4 battery	10 pcs prismatic	8 pcs 15 Ah prismatic pocket type cells				1.1 Ah 4 × 6 18650 lithium ion cells
Refer- ences	[55]	[56]	[57]	[58]	[59]	[36]	[60]	[61]	[62]	[63]	[64]
	34	35	36	37	38	39	40	41	45	43	4

**Table 1. Continuation** 

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Working output	The conical manifold design and opening a vent at the battery pack significantly increased cooling performance, significantly reducing fan power losses	By opening a vent at a point far from the cooling air outlet location, thermal management efficiency was increased and pumping power losses were reduced	The J-type thermal management system provided lower maximum temperature with extra air evacuation region, better temperature uniformity, thus better thermal management capability	The use of metal foam heat exchanger was found to be an effective method in terms of thermal management	The use of heat sink with aluminum foam heat transfer area provided better thermal management performance than non-aluminum foam heat sink	The use of metal foam in the heat absorber increased the cooling efficiency	Multiple inlets and outlets of cooling air improved battery thermal control performance	With the inclusion of porous material in the battery pack, the average temperature of the battery was reduced	This proposed porous cold plate structure was useful in preventing these damages that may occur during heating-cooling of the battery	Mass-flow rate of 5 g/s and water loading rate of 3% were sufficient to keep the maximum temperature and temperature difference of the battery module below 40 °C and 5 °C, respectively
T <sub>env</sub> [∘C]		27	22	>10	30	25				30
$\Delta T$ [°C]		$\stackrel{\scriptstyle <}{5}$			$\stackrel{\scriptstyle \wedge}{5}$	5	۲>			Ş
$T_{ m max}$ $[\circ  m C]$	52.8	<45	26.63		<35	40	<42		<35	<40
Battery load		5C	3C			3C	5C		5C	1- 3C
The way of work	Numerical (STAR CCM+)	Numerical	Numerical (ANSYS), experimental		Numerical (ANSYS-CFX)	Numerical (ANSYS)	Numerical (ANSYS)	Numerical (ANSYS)	Numerical, experimental	Numerical (ANSYS), experimental
Battery type and capacity	1400Wh 72 pcs cells	$12 \times 2$ pcs cells	1.6 Ah capacity LiMn <sub>2</sub> O <sub>4</sub> pocket type cells	50 Ah lithium titanate battery	3 pcs 20 Ah LiFePO <sub>4</sub> pocket type cell	19,5 Ah, 8 LiFePO4 cells	8S6P 18650 cell	40 cells battery		8 Ah, 8 lithium iron phosphate cells
Refer- ences	[65]	[99]	[67]	[68]	[69]	[70]	[71]	[72]	[73]	[74]
	45	46	47	48	49	50	51	52	53	54

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**Table 1. Continuation** 

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Working output	Type II and Type III are more advantageous at low inlet air velocities, while Type I is more advantageous at high inlet air velocities; therefore, it is not true that reverse air-flow will necessarily provide more advantageous thermal management	With this thermal management design, the battery pack has been enabled to operate within the safe range up to $48^\circ\text{C}$ ambient temperature	Developing a battery pack design that will shorten the air-flow path and increase the heat dissipation performance improves thermal management performance	Cylindrical cells provide less compact and more efficient thermal management capability than prismatic cells	Reduced heat build-up in the battery pack mid-cells with jet cooling; air inlet was provided from the bottom between the floor plate and the batteries, and heat removal between the batteries was strengthened	With the raised battery position, the flow distribution in the battery pack is increased homogeneously, thus providing higher thermal management capability	It was evaluated that the cooling air, which is used for cooling the battery system that meets the residential energy needs, is used as the heat source of the residential air conditioning system.	Cells of the battery pack were placed in a triangular arrangement. As the distance between the cells and the battery pack air outlets increases and the cooling air inlet speed increases, the battery pack temperatures decrease.	By adding six secondary air vents and a partition the ninth flow channel to the battery thermal management design, the max temperature and temperature difference were reduced by 4.95% and 91.89%, respectively.	Rectangular fins have been added on the cylindrical batteries so that the battery pack can onerate in the ontinuum temperature range.
$T_{ m env}$ $[\circ C]$		5-48	20	20	24	25		25	25	20, 30
$\Delta T$ [°C]						0.3			0.5	
$T_{ m max}$ $[\circ  m C]$		<60	0.6-2C		33.1			<30	41.65	<35
Battery load	5C				1C	1-10C				
The way of work	Numerical (Fluent 14.0), experimental	Numerical	Numerical	Numerical	Numerical, experimental	Numerical, experimental	Numerical (COMSOL)	Numerical (COMSOL)	Numerical, experimental	Numerical (FLUENT)
Battery type and capacity	24 pcs 18650, 26650, 42110 lithium-ion cells		55 Ah battery module				12 prismatic battery		15 Ah capacity	3200  mAh, $4 \times 6 \text{ cells}$
Refer- ences	[75]	[76]	[77]	[78]	[79]	[37]	[80]	[81]	[82]	[83]
	55	56	57	58	59	60	61	62	63	64

**Table 1. Continuation** 

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Summary information about many more studies is given in tab. 1. Fan *et al.* [60] evaluated the use of unequal channel spacing to improve temperature uniformity, as the cooling channel size increases between the cells at constant cooling air-flow rate, the maximum temperature difference decreases, the maximum temperature increases. It was stated that unequal channel spacing did not have a significant effect on improving temperature uniformity, that it has an lowering effect on maximum temperature. Li *et al.* [61] suggested using regionally different intercellular channel widths.

Zhang *et al.* [62] intercellular duct and cooling air inlet manifold evaluated to addition of flow diverter for to enhance the thermal management capability of the Z-type tangential blown air thermal management system. Research results showed that it effectively improves thermal management performance by increasing flow homogeneity of the addition of a flow diverter to the air distributor manifold and flow channels.

The U-type and Z-type tangential blower air thermal management systems create similar temperature distributions in opposite regions of the battery pack. The use of secondary vent is one of the preferred design initiatives in order to improve the maximum temperature and temperature homogeneity of U-type and Z-type tangential air discharge thermal management systems. Hong *et al.* [66]



evaluated the use of secondary vents to increase cooling efficiency. The diagram of the secondary vent locations evaluated in fig. 3. is given. Among the secondary vents, both the best thermal management efficiency and the lowest pumping power loss were achieved with the outlet -r, which is the vent at the furthest point from the cooling air outlet location.

Metal foams have become a preferred material for the development of heat transfer and flow processes in recent years, thanks to their advantageous geometries. Metal foam structure is also a preferred material in battery thermal management system design development studies [68-70] used a thermal air thermal management design with an aluminum foam-based heat exchanger. Giuliano *et al.* [68] used a thermal air thermal management design with an aluminum foam-based heat exchanger. Wang *et al.* [70] studied the effects of metal foam porosity and pore density on thermal management capability for a system using metal foam in the battery pack flow field. Wang *et al.* [70] stated that aluminum foam heat sink is used in thermal management system provides 30% lower average max temperature than without matal foam thermal management system, at the expense of pumping power losses. Removal of metal foam mass in certain areas of the heat sink reduced pumping power requirements by reducing pressure losses, improved temperature homogeneity significantly.

We have mentioned before, that use of air as the cooling fluid has limited thermal management performance due to the low specific heat capacity of the air. Saw *et al.* [74] proposed a highly original and remarkable design for the air thermal management system. It used mist air containing water droplets for battery thermal management. Using dry and mist air was compared experimentally and numerically. As a result of the research was determined that the system using mist cooling air provided better performance with maximum temperature drop and temperature uniformity. The reason for the decrease of the maximum temperature battery and the improvement of the thermal homogeneity battery is the evaporation of water droplets in mist air by absorbing the heat produced in the battery.

## Conclusions

Mobile robot systems have taken an important place with the development of battery power generation systems and computer systems in our lives. Temperature control is the biggest handicap of Li-ion battery power generation systems in order to operate safely within the required power, lifetime.

When the battery thermal management studies were examined, it was determined that the studies were mainly carried out on automotive applications, although real systems work with a lot of cells or battery module, almost all of the studies have been worked on a battery module or pack consisting of one or very few cells. Considering these situations, related studies should be interpreted for robot applications.

Thermal management studies were conducted on design proposition, design development and design optimization. The studies were mainly carried out as numerical simulation and experimental verification due to the long and costly experimental work. The results obtained in the studies are as follows.

- Regardless of the thermal management method, the thermal management of cylindrical cells is more efficient than prismatic cells due to their geometry. The uniformity of the thermal capacities of the battery pack cells increases the temperature homogeneity of the battery pack. In case of charge and discharge under the same conditions, thermal management loads are unequal, more heat is produced in the charging state than in the discharge.
- At medium and high charge-discharge loads, the use of without forced convection air convection for thermal management of the battery packet is insufficient in most cases. Thermal management performance is highly dependent on the placement of the cells in the battery pack. The battery pack array, which will reduce the interaction of the cells with each other as much as possible, provides better thermal management performance for air thermal thermal management without forced convection.
- Although forced air convection provides higher thermal management performance than non-forced convection, it is not a method that provides effective thermal management in all conditions. As with the without convection air thermal management system, the arrangement and arrangement of the batteries within the battery pack has an impact on thermal management performance. For forced convection air thermal management, the aligned arrays of cells in axial symmetry in the battery pack provide more effective thermal management performance. Another important consideration for a forced convection air thermal management system is the flow path of the cooling air through the battery pack. The cooling air progresses by losing its cooling efficiency due to the effect of heat gain and friction factor during the flow. For effective thermal management, the cooling air must be distributed homogeneously inside the battery pack and its travel distance must be short.
- In order to obtain a homogeneous temperature distribution in the battery pack, the cooling air must be homogeneously distributed in the battery pack. In the studies for this situation, using conical air inlet and outlet nozzles, adding a flow deflector to the battery pack and adding a secondary ventilation hole were the ways followed.

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