

VARIATIONAL PRINCIPLE FOR A GENERALIZED RABINOWITSCH LUBRICATION

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This paper adopts Rotem and Shinnar's modification of the Rabinowitsch fluid model for the 1-D non-Newtonian lubrication problem, a variational principle is established by the semi-inverse method, and a generalized Reynolds-type equation is obtained. This article opens a new avenue for the establishment of Reynolds-type equation of complex lubrication problems.

Key words: *Lagrange functional, variational theory, Euler-Lagrange equation, Reynolds equation, He's semi-inverse method*

Introduction

Non-Newtonian fluids appear everywhere in engineering applications, for examples, polymer solutions [1, 2], nanofluids [3, 4], Walters' B fluid [5], Reiner-Rivlin fluid [6], phase change material [7], and peristaltic flow [8]. Zuo [9] suggested a fractal rheological model for non-Newtonian fluids, and Liang and Wang [10] gave a fractal viscoelastic element for various non-Newtonian fluids.

The Rabinowitsch fluid model is widely used in the non-Newtonian lubrication theory [11-18]. This paper considers a 1-D slide bearing as illustrated in fig. 1.

The Rabinowitsch 1929 model assumes the following constitutive relationship [19]:

$$\sigma_{xz} + a\sigma_{xz}^3 = \eta \frac{\partial u}{\partial z} \quad (1)$$

where σ_{xz} is the shear stress, u – the velocity, η – the viscosity coefficient, and a – the Rabinowitsch parameter for non-Newtonian property.

A more generalized modification of eq. (1) was suggested by Rotem and Shinnar in the form [20]:

$$\sigma_{xz} + \sum_{n=1}^N a_n \sigma_{xz}^{2n+1} = \eta \frac{\partial u}{\partial z} \quad (2)$$

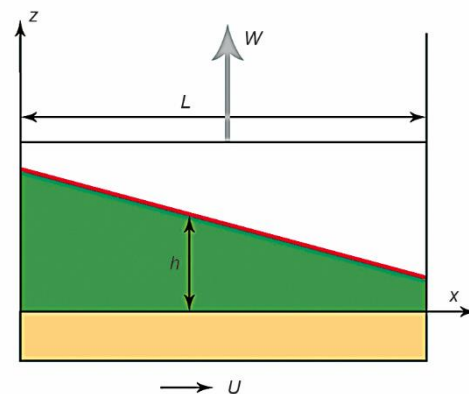


Figure 1. Geometric structure of a sliding surface

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where $a_n (n=1 \sim N)$ are constants. In this paper we adopt the following one with a cubic-quinary non-linearity:

$$\sigma_{xz} + a_1 \sigma_{xz}^3 + a_2 \sigma_{xz}^5 = \eta \frac{\partial u}{\partial z} \quad (3)$$

where a_1 and a_2 are Rabinowitsch parameters.

Variational formulation

The governing equations for 1-D non-Newtonian lubrication can be expressed [18]:

$$\frac{\partial u}{\partial x} + \frac{\partial w}{\partial z} = 0 \quad (4)$$

$$\frac{\partial p}{\partial x} = \frac{\partial \sigma_{xz}}{\partial z} \quad (5)$$

$$\frac{\partial p}{\partial z} = 0 \quad (6)$$

where u , w are velocities in x - and z -directions, respectively, and p – the pressure.

Equation (1) is for the mass conservation, while eqs. (5) and (6) are the moment conservation in x - and z -directions, respectively.

The boundary conditions are:

$$z = 0: u = U, \quad w = 0 \quad (7)$$

$$z = h: u = 0, \quad w = W \quad (8)$$

In this section, we search for a variational formulation for the aforementioned lubrication problem. The variational principle is widely used in engineering to establish a needed differential model for practical problems. Various variational formulations were appeared in literature for various problems, for examples, water waves [21-24], non-linear vibration system [25, 26], Burgers equation [27], Benney-Lin equation [28], solitary waves [29-31]. In this paper, the semi-inverse method [32-34] is applied to establish a needed variational formulation.

According to the semi-inverse method [32-34], we construct the following trial-functional:

$$J(u, w, p, \sigma_{xz}) = \int_0^L \int_0^h \left[u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} + F(u, w, \sigma_{xz}) \right] dx dz \quad (9)$$

where F is unknown yet, it will be determined step by step. Alternative approaches to construction of the trial-functional are available in [35, 36].

The stationary condition with respect to p is eq. (4), and we obtain the following two equations with respect to u and w , respectively:

$$\frac{\partial p}{\partial x} + \frac{\delta F}{\delta u} = 0 \quad (10)$$

$$\frac{\partial p}{\partial z} + \frac{\delta F}{\delta w} = 0 \quad (11)$$

where $\delta F/\delta u$ is the variational derivative defined:

$$\frac{\delta F}{\delta u} = \frac{\partial F}{\partial u} - \frac{\partial}{\partial x} \left(\frac{\partial F}{\partial u_x} \right) - \frac{\partial}{\partial z} \left(\frac{\partial F}{\partial u_z} \right) \quad (12)$$

In view of eqs. (5) and (6), we can convert eqs. (10) and (11) into the following forms:

$$\frac{\delta F}{\delta u} = -\frac{\partial p}{\partial x} = -\frac{\partial \sigma_{xz}}{\partial z} \quad (13)$$

$$\frac{\delta F}{\delta w} = -\frac{\partial p}{\partial z} = 0 \quad (14)$$

We, therefore, can determine F :

$$F = -u \frac{\partial \sigma_{xz}}{\partial z} + f(\sigma_{xz}) \quad (15)$$

where f is unknown yet. Now eq. (9) becomes:

$$J(u, w, p, \sigma_{xz}) = \int_0^L \int_0^h \left[u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} - u \frac{\partial \sigma_{xz}}{\partial z} + f(\sigma_{xz}) \right] dx dz \quad (16)$$

Now the stationary condition with respect to σ_{xz} is:

$$\frac{\partial u}{\partial z} + \frac{\delta f}{\delta \sigma_{xz}} = 0 \quad (17)$$

By eq. (3), we have:

$$\frac{\delta f}{\delta \sigma_{xz}} = -\frac{\partial u}{\partial z} = -\frac{1}{\eta} (\sigma_{xz} + a_1 \sigma_{xz}^3 + a_2 \sigma_{xz}^5) \quad (18)$$

From eq. (18), we have:

$$f = -\frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \quad (19)$$

Equation (16) is updated:

$$J(u, w, p, \sigma_{xz}) = \int_0^L \int_0^h \left[u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} - u \frac{\partial \sigma_{xz}}{\partial z} - \frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \right] dx dz \quad (20)$$

By a similar derivation as given in [18], the boundary integral can be incorporated. We assume that:

$$J(u, w, p, \sigma_{xz}) = \int_0^L \int_0^h \left[u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} - u \frac{\partial \sigma_{xz}}{\partial z} - \frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \right] dx dz + \int_0^L g(p, \sigma_{xz}) dx \quad (21)$$

where g is unknown. The natural conditions are:

$$u|_{x=0}^{x=h} + \frac{\delta g}{\delta p} = 0 \quad (22)$$

$$-u|_{x=0}^{x=h} + \frac{\delta g}{\delta \sigma_{xz}} = 0 \quad (23)$$

In view of the boundary conditions of eqs. (7) and (8), we have:

$$\frac{\delta g}{\delta p} = -u|_{x=0}^{x=h} = -[W(h) - W(0)] \quad (24)$$

$$\frac{\delta g}{\delta \sigma_{xz}} = u|_{x=0}^{x=h} = U \quad (25)$$

From eqs. (24) and (25), we obtain:

$$g = -[W(h) - W(0)]p + U\sigma_{xz} \quad (26)$$

We finally obtain the following generalized variational principle:

$$J(u, w, p, \sigma_{xz}) = \int_0^L \int_0^h \left[u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} - u \frac{\partial \sigma_{xz}}{\partial z} - \frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \right] dx dz - \\ - \int_0^L [W(h) - W(0)] p dx + \int_0^L U \sigma_{xz} dx \quad (27)$$

Proof. The Lagrange function of eq. (27) is:

$$\lambda = u \frac{\partial p}{\partial x} + w \frac{\partial p}{\partial z} - u \frac{\partial \sigma_{xz}}{\partial z} - \frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \quad (28)$$

and the stationary conditions of eq. (27) are:

$$\frac{\partial \lambda}{\partial \xi} - \frac{\partial}{\partial x} \left(\frac{\partial \lambda}{\partial \xi_x} \right) - \frac{\partial}{\partial z} \left(\frac{\partial \lambda}{\partial \xi_z} \right) = 0 \quad (29)$$

where ξ implies to u , or w , or p or σ_{xz} , and its subscription implies the partial derivative, *e.g.* $\xi_x = \partial \xi / \partial x$. The stationary conditions with respect to u , w , p , and σ_{xz} are given, respectively:

$$-\frac{\partial \sigma_{xz}}{\partial z} + \frac{\partial p}{\partial x} = 0 \quad (30)$$

$$\frac{\partial p}{\partial z} = 0 \quad (31)$$

$$-\frac{\partial u}{\partial x} - \frac{\partial w}{\partial z} = 0 \quad (32)$$

$$-\frac{1}{\eta}(\sigma_{xz} + a_1\sigma_{xz}^3 + a_2\sigma_{xz}^5) + \frac{\partial u}{\partial z} = 0 \quad (33)$$

It is obvious that eqs. (30)-(33) are the governing equations.

Reynolds-type equation

Similar to the derivation as given in [18], we obtain the following constrained variational principle:

$$J(\sigma_{xz}) = \int_0^L \int_0^h \left[-\frac{1}{\eta} \left(\frac{1}{2} \sigma_{xz}^2 + \frac{1}{4} a_1 \sigma_{xz}^4 + \frac{1}{6} a_2 \sigma_{xz}^6 \right) \right] dx dz - \int_0^L [W(h) - W(0)] p dx + \int_0^L U \sigma_{xz} dx \quad (34)$$

which is subject to eqs. (3), (4), and (6).

Integrating eq. (5) with respect to z and identifying the integration constant, we have:

$$\sigma_{xz} = \left(z - \frac{h}{2} \right) \frac{\partial p}{\partial x} \quad (35)$$

Substituting eq. (35) into eq. (34), and integrating from $z=0$ to $z=h$, we have:

$$J(p) = \int_0^L \left\{ -\frac{1}{\eta} \left[\frac{1}{24} h^3 \left(\frac{\partial p}{\partial x} \right)^2 + \frac{1}{320} a_1 h^5 \left(\frac{\partial p}{\partial x} \right)^4 + \frac{1}{2680} a_2 h^7 \left(\frac{\partial p}{\partial x} \right)^6 \right] \right\} dx - \int_0^L [W(h) - W(0)] p dx + \int_0^L \frac{1}{2} U h \frac{\partial p}{\partial x} dx \quad (36)$$

Now the stationary condition of eq. (36) is:

$$\frac{\partial p}{\partial x} \left\{ \frac{1}{\eta} \left[\frac{1}{12} h^3 \frac{\partial p}{\partial x} + \frac{1}{80} a_1 h^5 \left(\frac{\partial p}{\partial x} \right)^3 + \frac{1}{448} a_2 h^7 \left(\frac{\partial p}{\partial x} \right)^5 \right] \right\} - [W(h) - W(0)] - \frac{\partial}{\partial x} \left(\frac{1}{2} U h \right) = 0 \quad (37)$$

Note that:

$$W(h) - W(0) = \frac{\partial h}{\partial t} \quad (38)$$

Finally we obtain the following Reynolds-type equation:

$$\frac{\partial p}{\partial x} \left\{ \frac{1}{\eta} \left[\frac{1}{12} h^3 \frac{\partial p}{\partial x} + \frac{1}{80} a_1 h^5 \left(\frac{\partial p}{\partial x} \right)^3 + \frac{1}{448} a_2 h^7 \left(\frac{\partial p}{\partial x} \right)^5 \right] \right\} = \frac{\partial h}{\partial t} + \frac{\partial}{\partial x} \left(\frac{1}{2} U h \right) \quad (39)$$

Discussion and conclusion

In case $a_2 = 0$, eq. (37) reduces to:

$$\frac{\partial p}{\partial x} \left\{ \frac{1}{\eta} \left[\frac{1}{12} h^3 \frac{\partial p}{\partial x} + \frac{1}{80} a_1 h^5 \left(\frac{\partial p}{\partial x} \right)^3 \right] \right\} = \frac{\partial h}{\partial t} + \frac{\partial}{\partial x} \left(\frac{1}{2} U h \right) \quad (40)$$

This is same as those in [14, 18]. When $a_1 = a_2 = 0$, eq. (37) reduces to the classic Reynolds equation [14, 18], which is:

$$\frac{\partial p}{\partial x} \left\{ \frac{1}{\eta} \left[\frac{1}{12} h^3 \frac{\partial p}{\partial x} + \frac{1}{80} a_1 h^5 \left(\frac{\partial p}{\partial x} \right)^3 \right] \right\} = \frac{\partial h}{\partial t} + \frac{\partial}{\partial x} \left(\frac{1}{2} U h \right) \quad (41)$$

For the modification of eq. (2) given by Rotem *et al.* [20], the Reynolds-type equation can be derived by a similar manner discussed above, and it reads:

$$\frac{\partial p}{\partial x} \left\{ \frac{1}{\eta} \left[\frac{1}{12} h^3 \frac{\partial p}{\partial x} + \sum_{n=1}^N \frac{h^{2n+3}}{2^{2n+2} (2n+3)} a_n \sigma_{xz}^{2n+2} \right] \right\} = \frac{\partial h}{\partial t} + \frac{\partial}{\partial x} \left(\frac{1}{2} U h \right) \quad (42)$$

Lubrication arises everywhere including many food processes, and the present theoretical analysis is helpful to establish a mathematical model and design an optimal lubrication system.

To be concluded, this paper suggests a new approach to the establishment of a Reynolds-type equation for a complex lubrication problem.

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