A SEMI-ANALYTICAL SOLUTION OF 3-D TRANSIENT TEMPERATURE FIELD FOR A UNIFORM PLATE SUBJECTED TO GAUSSIAN-DISTRIBUTION LASER HEAT SOURCE

by

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One 3-D transient temperature field model for a thin uniform plate caused by a moving laser heat source is described in present study. The heat source model with a power density in Gaussian-distribution form is considered when a finite-thin uniform plate is heated. By using the separate variable method and the Newton-Cotes method, a semi-analytical solution of 3-D heat conduction equation in the finite field is obtained. Numerical results show that the effect of laser heat source distribution, laser moving speed as well as aspect ratio of the thin uniform plate have great influence on the 3-D distribution of the temperature field.

Key words: 3-D temperature distribution, Gaussian-distribution heat source, separate variable method, Newton-Cotes method

Introduction

Lasers were widely used [1-5] in various processing and manufacturing fields due to their features such as high efficiency, easy power adjustment, and no force during processing. The temperature of the material was increased by the heat of the laser [6]. When the temperature rises to a certain degree, the material would break, melt, and deform. Therefore, it is particularly important to perform temperature prediction and controlling during laser operations to avoid dangerous thermal stresses.

There were some problems in the practical application of laser assisted processing. For instance, determining the optimal process parameters of the laser [7-9], the interaction mechanism of the laser and the material (such as the absorptivity of the material to the laser [10-12], the penetration ability of the laser to the material [13, 14], and the phase change of the laser to the material influences [15-18], *etc.*). Therefore, a large number of experimental studies were conducted. Experiments [19-23] were conducted to evaluate the potential of laser-assisted processing in the economic feasibility of manufacturing precision ceramic components. Comparing the surface damage of parts produced by laser-assisted processing with the damage of typical grinding parts [24], the results showed that the most significant advantages of laser-

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assisted processing could achieve higher material removal rate, more excellent workpiece surface quality, and well reasonable tool wear levels. Accurately measuring the temperature field of a workpiece during laser processing was extremely difficult, and it often required many previous tests to set the suitable laser process [25] for certain materials, this would require a lot of manpower, material, and financial resources. Besides, human error in operation and the advancement of measuring instruments all affect the final experimental results.

With the rapid development of computer software and hardware technology, virtual manufacturing technology has caused an upsurge, and numerical simulation has gradually behaved its superiority [26]. In terms of numerical simulation, regarding the laser source as a concentrated heat source, Gutierrez and Araya [27] developed a numerical simulation of the temperature distribution generated by a moving laser beam for a finite field. Comparing with the analytical solution in the semi-infinite field [28], slightly higher temperatures were produced in the regions that close to the boundary. The two temperature profiles actually coincided at that far away from the border. By using finite element analysis software (ANSYS), Han et al. [29] simulated the temperature distribution of the laser welding of the 304 stainless steel plate dynamically. The calculation showed that the simulation results of the weld shape agreed well with the experimental results. Considering the effects of phase transition potential [30, 31] as well as thermal, structural and plastic deformations, Yao and Chen [32] presented a 3-D finite element model for the calculation of transient thermal and residual stresses, besides, the numerical simulation of the thermal field and residual stress field during laser processing was studied. The numerical example verified that the current model was better than other models that had been proposed, but there also existed errors comparing with the experimental results. The finite element model was used to simulate the sintering process of Ni-Gr coatings on AISI 4140 steel [33] numerically, and the distribution and evolution of its temperature field was also studied, single-track experiments were performed with the same laser sintering parameters as the simulations, the results showed the correctness of the simulation. Li et al. [34] developed a 3-D thermo mechanical coupling model to simulate a multi-track, multi-layered selective laser melting process by using the finite element method. The numerical solution was generally a solution obtained using a finite element method, a numerical approximation method, an interpolation method and so on. However, the calculated values corresponding to any independent variables as well as the value of the independent variables subject to certain restrictions of the numerical method could not be obtained. Thus, the semi-analytical solutions were often introduced to solve these problems effectively.

In terms of theoretical calculations, Elsen *et al.* [35] studied the analysis and numerical solution of the heat conduction equation for any type of local moving heat source for laser shock processing. Cheng and Lin [36] established an analytical model that describing a 3-D temperature field that using a Gaussian heat source moving at a constant velocity, and the computation time of analytical solution cost less when comparing with the finite element method and the finite difference method. Considering the cooling effect caused by free or forced convection in the ambient gas, Brockman *et al.* [37] calculated a 2-D temperature field in a sheet moving slice heated with a laser beam, and a 2-D integral Fourier transform on the space coordinates was used by them to solve the problem as well as the analytical solution of the problem in Fourier space was obtained. Jiang and Dai [38] solved the 3-D steady heat conduction problem of a double-layer plate by using the Poisson method and the layer wise approach, where the double-layer structure including a coating layer and functionally graded material layer, in addition, the separation of variables method and the layer wise approach were used to solve the 3-D transient thermal conduction problem. Taking into consideration the solid-liquid-vapor phase change process [39] and the simply supported conditions [40], respectively, Jiang *et al.* [39] solved the 3-D transient temperature field of a high strength and low alloy rectangular steel plate under laser shock processing by the separate variable method (SVM). Considering the trajectory of the source was approximately a straight line as well as surface heat source model was regarded as a Gaussian-distribution of power density, Winczek *et al.* [41] described a 3-D transient temperature field in a semi-infinite body caused by a moving laser heat source with any trajectory. Chen *et al.* [42] gave a short review of the harmonic general solutions for uncoupled elasticity of transversely isotropic materials with thermal and other effects.

Most of analytical solutions encountered in the literatures were based on an infinite domain, semi-infinity domain as well as quasi-steady state at a constant speed for heat source. Few works referred to complete semi-analytical solutions for 3-D transient heat conduction, and only the temperature evolution caused by a uniform heat source was described. Present paper focus on the Gaussian-distribution of laser heat source for temperature model in the uniform plate firstly. Then, the result of paper is compared with that of numerical simulation and shows the accuracy of the present method. Finally, the effects of heat source distribution, laser moving speed and the aspect ratio of length to width for the uniform plate on the distribution of the temperature field is analyzed.

Mathematical modeling

Figure 1 is a Si₃N₄ plate (whose dimensions in *x*-, *y*-, and *z*-directions are *a*, *b*, and *h*, respectively) subjected to moving laser beam source with uniform velocity, *v*, along the *x*-direction. The heat from laser source penetrates into the plate and generates a continuous heat flux, *q*, going into the plate with the passage of time, *t*, thus the heat conduction model could be regarded as under the transient thermal environment. The following assumptions are also considered in the present model [43]:

- The investigated Si₃N₄ plate is uniform and isotropic.
- The physical properties such as thermal conductivity, λ, density, ρ, and specific heat, c, of the structure are all constants.



Figure 1. Schematic of the laser processing on a uniform Si₃N₄ plate

Temperature jump or phase change does not occur in the temperature field for the plate.
 With the previous assumptions and considering 3-D heat conduction process, the tran-

sient heat conduction equation based on the energy conservation law can be written:

$$oc \frac{\partial \Delta T}{\partial t} = \lambda_x \frac{\partial^2 \Delta T}{\partial x^2} + \lambda_y \frac{\partial^2 \Delta T}{\partial y^2} + \lambda_z \frac{\partial^2 \Delta T}{\partial z^2} + q'''(x, y, z, t)$$
(1)

where $\lambda_i (i = x, y, z)$ are thermal conductivities along *x*-, *y*-, and *z*-directions, respectively, $\Delta T = T - T_0$ is the temperature variation for the uniform plate relative to surrounding environment temperature, and q'''(x, y, z, t) is the volumetric source term of heat generation.

The 3-D heat source modeling

The heat source term q''(x, y, z, t) is considered as a laser beam of temporal continuous wave and spatially modeled as a Gaussian-distribution. Figure 2 shows the transversal mode in a laser spot.

The Gaussian heat flux distribution, q'', takes the following exponent form:

$$q'' = \begin{cases} I_0 e^{\frac{-k(x^2+y^2)}{r^2}} vt - r < x < vt + r, & -\sqrt{r^2 - (x - vt)^2} < y < \sqrt{r^2 - (x - vt)^2}, & \text{and} & z = 0 \\ 0 & \text{otherwise} \end{cases}$$
 (2)



Figure 2. Transversal mode in a laser spot

where I_0 is the intensity at the center of the spot, k - a constant (in general, a value of 2 is used for a Gaussian model), and r – the laser spot radius, with following form:

$$r = \sqrt{\frac{s}{\pi}} \tag{3}$$

where s is the laser spot area, besides heat flow absorbed by the workpiece. The q can be obtained by following expression:

 $\int_{-r}^{r} \int_{\sqrt{r^2 - x^2}}^{\sqrt{r^2 - x^2}} I_0 e^{\frac{-k(x^2 + y^2)}{r^2}} dy dx = q$ (4)

The relation between volumetric heat generation, q'', and Gaussian-distribution heat flux, q'', is defined:

$$q''' = q''\delta(z) \tag{5}$$

where $\delta(z)$ is the Dirac delta function.

Initial and boundary conditions

Boundary conditions at the six faces of the parallelepiped domain and an initial condition have to be specified. The initial condition of the temperature variation, ΔT , for the Si₃N₄ plate is as follows when the time equals to zero:

$$\Delta T(x, y, z, 0) = 0 \tag{6}$$

Insulated conditions were imposed at x = 0 surface, *i. e.* with:

$$\Delta T(0, y, z, t) = 0 \tag{7}$$

Meanwhile, Dirichlet temperature conditions were considered at x = a, y = b/2, y = -b/2, and z = h surfaces. The main reason of selecting Dirichlet conditions at mentioned surfaces can be seen in reference [43]. Besides, the top surface, z = 0, is considered to be adiabatic except for the heating area which powered by the moving laser beam.

In order to solve the 3-D transient temperature filed subjected to Gaussian-distribution heat flux, the temperature field $\Delta T(x, y, z, t)$ in eq. (1) can be expressed by using the (SVM):

$$\Delta T(x, y, z, t) = \sum_{o} \sum_{p} \sum_{q} \Theta(o, p, q, t) X(o, x) Y(p, y) Z(q, z)$$
(8)

According to the initial and boundary conditions, *i. e.* eqs. (6) and (7), respectively, the eigenfunctions X(o, x), Y(p, y), and Z(q, z), which satisfied mentioned boundary conditions have the following forms, respectively:

$$X(o, x) = \cos[\alpha(o)x]$$
(9a)

$$Y(p, y) = \cos[\beta(p)y]$$
(9b)

$$Z(q,z) = \cos[\gamma(q)z]$$
(9c)

where the eigenvalues $\alpha(o)$, $\beta(p)$, and $\gamma(q)$, are, respectively:

$$\alpha(o) = \frac{2o+1}{2a}\pi, \quad o = 0, 1, \dots n$$
 (10a)

$$\beta(p) = \frac{2p+1}{b}\pi, \quad p = 0, 1, \dots n$$
 (10b)

$$\gamma(q) = \frac{2q+1}{2h}\pi, \quad q = 0, 1, \dots n$$
 (10c)

The non-homogeneous term, q'', can be expanded in a linear combination of the eigenfunctions:

$$q'''(x, y, z, t) = \sum_{o} \sum_{p} \sum_{q} \varphi(o, p, q, t) X(o, x), Y(p, y) Z(q, z)$$
(11)

By utilizing the orthogonality of eigenfunctions X(o, x), Y(p, y), and Z(q, z), and multiplying them side-by-side for eq. (11) as well as integrating over the whole dimensional domain, then eq. (11) yields:

$$\int_{0}^{a} \int_{-b/2}^{b/2} \int_{0}^{h} q'''(x, y, z, t) X(o, x) Y(p, y) Z(q, z) \, dx dy dz =$$

$$= \varphi(o, p, q, t) \int_{0}^{a} X^{2}(o, x) dx \int_{-b/2}^{b/2} Y^{2}(p, y) dy \int_{0}^{h} Z^{2}(q, z) dz =$$

$$= \varphi(o, p, q, t) \frac{abh}{8}$$
(12)

Substituting eqs. (2) and (5) into eq. (12) and integrating it into the entire field, following equation can also be obtained:

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$$\int_{0}^{a} \int_{-b/2}^{b/2} \int_{0}^{h} I_{0} e^{\frac{-k(x^{2}+y^{2})}{r^{2}}} X(o,x) Y(p,y) Z(q,z) \delta(z) \, dz \, dy \, dx =$$

$$= \int_{vt-r}^{vt+r} \int_{-\sqrt{r^{2}-(x-vt)^{2}}}^{\sqrt{r^{2}-(x-vt)^{2}}} I_{0} e^{\frac{-k[(x-vt)^{2}+y^{2}]}{r^{2}}} \cos[\alpha(o)x] \cos[\beta(p)y] \, dx \, dy =$$

$$= \varphi(o, p, q, t) \frac{abh}{8}$$
(13)

Numeral calculations

It is so hard to solve eq. (13) through direct integration, the Newton-Cotes method (NCM) is considered on left side of eq. (13), the NCM has following unified expression form:

$$C_m(f) = \sum_{i=0}^{m-1} \frac{l}{90} [7f(x_i) + 32f(x_{i+1/4}) + 12f(x_{i+1/2}) + 32f(x_{i+3/4}) + 7f(x_{i+1})]$$
(14)

where *C* is the abbreviation of NCM, subscript m – number of divided segments, f – the function to be integrated, and l – the length of each segment.

The following equation should be satisfied for a given minimum accuracy, ε :

$$|I(f) - C_{2m}(f)| \approx \frac{1}{63} |C_{2m}(f) - C_m(f)| \le \varepsilon$$
 (15)

where I(f) represents the exact value of the quadrature function, f. In this paper, the value of m is taken as 8, which not only satisfies certain accuracy requirements, but also saves the calculation time.

Finally, the expression of $\varphi(o, p, q, t)$ can be obtained according to eqs. (13) and (14).

Substituting the $\varphi(o, p, q, t)$ into eq. (11) and inserting it into eq. (1) together with eq. (8), then an ODE for $\Theta(o, p, q, t)$ can be obtained:

$$\frac{1}{\alpha}\frac{\mathrm{d}\Theta(o,p,q,t)}{\mathrm{d}t} = -\frac{\omega(o,p,q)}{\alpha}\Theta(o,p,q,t) + \frac{\varphi(o,p,q,t)}{\lambda} \tag{16}$$

where

$$\alpha = \frac{1}{\rho c}, \quad \omega(o, p, q) = \{ [\alpha(o)]^2 + [\beta(p)]^2 + [\gamma(q)]^2 \} \alpha$$
(17)

Defining $\Omega(o, p, q, t) = (a/\lambda)\varphi(o, p, q, t)$, then eq. (16) yields:

$$\frac{\mathrm{d}\Theta(o, p, q, t)}{\mathrm{d}t} + \omega(o, p, q)\Theta(o, p, q, t) = \Omega(o, p, q, t)$$
(18)

With the initial condition eq. (6), there with $\Delta T = 0$ when t = 0. The corresponding X(o, x), Y(p, y), and Z(q, z) values are not always 0 for any x, y, or z, hence the following equation can be finally obtained:

$$\Theta(o, p, q, 0) = 0 \tag{19}$$

The solution of $\Theta(o, p, q, t)$ can be obtained according to simultaneous eqs. (18) and (19). Then substituting the expression of $\Theta(o, p, q, t)$ into eq. (8), the transient temperature field can be finally obtained.

Results and discussion

The semi-analytical solution is used for the laser heating process of the uniform Si₃N₄ plate, whose thermophysical properties [43-45] are summarized in tab.1. The area of the laser circular spot $1 \cdot 10^{-6}$ m² and the laser net power 50 W are considered in present work. From the established Cartesian co-ordinate system, fig. 1, the temperature distribution on the workpiece is symmetrical about the *x* axis, hence considering half of the workpiece is enough. Referring to [43], it is determined that the 150 feature value is taken in each direction.

Tuble 1. Thermophysical properties of smeon mariae (Dista	Τa	able	e 1.	Therm	ophysical	l pro	perties	of	silicon	nitride	(Si ₃	3N4)
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Melting point	Density	Specific heat	Thermal conductivity	Thermal diffusivity
[°C]	[kgm ⁻³]	[Jkg ⁻¹ °C ⁻¹] at 1000 °C	[Wm ⁻¹ °C ⁻¹] at 1000 °C	[m ² s ⁻¹] at 1000 °C
1900	2400	1278.96	5.91	$1.9 \cdot 10^{-6}$

Example 1. The dimensions of the workpiece along the *x*-, *y*-, and *z*-directions are 0.04 m, 0.02 m, and 0.00625 m, respectively. The laser moving speed 0.1 m/s is considered, besides the k value of Gaussian heat source is se-

lected as 3. When the laser moves to 0.25 a at *x*-direction, the temperature distribution along the *x*-direction of workpiece is shown in fig. 3.

From fig. 3, the current semi-analytical solution is compared with numerical solution simulated by Abaqus commercial software (also in present work) and the results of [43]. The three temperature curves are approximately the same, which indicates the correctness of the semi-analytical solution and numerical solution. There is a preheat zone in front of the laser spot due to thermal conduction. The temperature gradient near the area of laser heat source is very steep, and it drops to room temperature quickly far from the laser spot. The peak temperatures of semi-analytical solution, literature, and numerical simulation are 2259 °C, 2178 °C, and



Figure 3. Comparison of the semi-analytical solution with numerical simulation and [43] along the *x*-direction

2134 °C, respectively. The peak temperature of the simulation and the calculation time are closely related to the division of the grid. Hence, it is important to select the reasonable partition grid.

Example 2. Only changing the laser moving time, and the corresponding temperature distribution are obtained. Here, the *x*-direction grid is divided evenly, and the grids in the *y*- and *z*-directions are divided proportionally (the grid near the laser region is denser, and the grid farther away from the laser region is sparse).

Figure 4 shows the typical results of temperature distribution for uniform plate at various moving distances (0.1, 0.2, 0.25, and 0.3 seconds location). The shape of the temperature distribution for workpiece surface is also of a Gaussian form as the Gaussian-distribution laser source is considered.



Figure 4. The 3-D temperature distribution on the workpiece surface at different moving distances *(for color image see journal web site)*

The comparison between the semi-analytical solution and the numerical solution simulated by Abaqus commercial software are shown in figs. 5(a)-5(c). From fig. 5(a)-5(c), the temperature profiles of the workpiece along the *x*-, *y*-, and *z*-directions are well consistent when the heat source is located at x = 0.75a. The peak values of temperature for the semi-analytical solution and the numerical simulation are 2260 °C and 2188 °C, respectively. Compared with fig. 3, the peak temperature of fig. 5(a) increases slightly with the increasing of the laser moving distance. Furthermore, from fig. 5, it is noteworthy that the peak values of temperature for both curves lag behind the laser center.



Figure 5. Comparison of the semi-analytical solution with numerical simulation along the (a) x-, (b) y-, and (c) z-directions, respectively

Figures 5(b) and 5(c) show the temperature distribution along the y- (z = 0 surface) and z- (y = 0 location) directions of the uniform plate, respectively. As shown in the both two figures, the farther away from the center of the spot is, the lower is the temperature. Besides, it can be found that the temperature values along the y-axis and z-axis reach room temperature at 0.07 b/2 and 0.06 h, respectively. That is, the temperature gradient along the y-axis decreases more moderately as well as distributed more evenly.



Figure 6. Temperature profiles along the (a) *y*- and (b) *z*-directions, of which over and behind the laser spot position

Figures 6(a) and 6(b) show the trends of the temperature profile for the uniform plate along the y- (z = 0 surface) and z- (y = 0 location) directions, respectively, where z remains unchanged, only the different x locations is considered. From fig. 6(a), the temperature change in the y-direction is smooth at the laser center, but the change in the z-direction is a little sharp

from fig. 6(b). The temperature value at the laser center drops rapidly as it can be seen from the both two figures that the temperature value at x = 0.75a is approximately 3.5 times than that at x = 0.7a location.

Example 3. This example mainly studies the effect of the heat source model on the temperature distribution for uniform plate. Herein, the size of the workpiece as well as speed of laser do not change, then the temperature distribution along the *x*-direction of uniform plate when the laser moves to 0.25 a (y = 0, z = 0) with different k values is considered.

Figure 7 shows that the larger the *k*-value is, the thinner and higher is the energy density distribution curve as well as the steeper for the



Figure 7. Effect of heat source model *k* on temperature distribution along the *x*-direction

temperature gradient near the core area of laser source. For different heat source models, the temperature distributions in the preheating zone (y < 0.2 a) and the tail zone (x > 0.3 a) are almost the same, besides the size of heat affected zone size along the *x*-direction is also almost the same.

Example 4. Figures 8(a)-8(c) show effect of laser moving speed on temperature distribution for the uniform plate. Herein, the *k* value is selected as 3 and the laser source is located at x = 0.75a position. The laser moving speed affects the overlap rate of spot during laser processing directly, hence increasing the laser moving speed can reduce the heat accumulation effect during the process. From fig. 8(a), the laser moving speed cannot change the time of energy conduction, but it has a great influence on the temperature change of the material's heat affected zone, while effect on the temperature of the preheating zone on the surface of the uniform plate is too little and can be ignored.



Figure 8. Effect of laser moving speed on temperature distribution along the (a) *x*-, (b) *y*-, and (c) *z*-directions

From fig. 8(b), the temperature gradient in the y-direction decreases immediately, and the maximum surface temperature is higher due to the limit of area for workpiece as well as continuous laser heating on the surface of plate. The farther from the center of the spot radius it is, the lower is the temperature for the uniform plate and decays in exponential form. Besides, the temperature gradient changes rapidly due to the heating effect of the laser beam on the workpiece surface is just in the laser spot radiation area, while the workpiece is only affected by thermal radiation on the non-radiation area of the laser heat source, hence the temperature gradient change at this area is slower when compared with the heated surface.

Figure 8(c) shows that the temperature as well as deposition of energy decrease with the increasing of laser moving speed at the laser source area, and the larger z becomes, the lower the temperature is. The cutting tool should be as close to the spot as possible to achieve the purpose of processing when laser-assisted processing is performed as the thickness of the affected workpiece is limited. Besides, from figs. 8(a)-8(c), it can also be observed that the



Figure 9. Effect of aspect ratio *a/b* on temperature distribution along the (a) *x*- and (b) *z*-directions

heat-affected zones in the three directions of x, y, and z decrease with the increasing of laser moving speed.

Example 5. The effect of different aspect ratios on the temperature distribution is studied, and the laser speed is selected as 0.1 m/s in this example. The length and height of the workpiece are not changed, besides the length-width ratio 2, 5, and 10 are considered, respectively. Figures 9(a) and 9(b) demonstrate effect of aspect ratio a/b on temperature distribution along the x-direction (y = 0, z = 0) and z-direction (y = 0), respectively. With the change of the aspect ratio a/b, the temperature distribution in the area that away from the light spot does not change, and only the tiny area at the laser spot has a slight influence for the temperature. Therefore, it can be concluded that changing the size of the workpiece is not a feasible method to control the temperature distribution.

Conclusions

In this study, one semi-analytical solution for describing the 3-D transient temperature distribution caused by the moving laser heat source in the finite uniform plate is determined. The effect of Gaussian-distribution heat source parameters, k, laser moving speed, and workpiece aspect ratio, a/b, on the temperature distribution are analyzed systematically.

The results show that the peak temperature lags behind the center of the laser and the temperature changes very steeply around the center of the laser source. The temperature in the preheating zone increases while the temperature in the tail zone remains constant essentially. The larger the k value is, the more concentrated is the energy density distribution in the laser spot as well as and the higher for the maximum temperature of the workpiece. The smaller the laser moving speed is, the more energy is deposited into the workpiece as well as the higher for the workpiece temperature. The aspect ratio of the workpiece has no significant effect on the temperature distribution, thus adjusting the temperature distribution by changing the aspect ratio of the workpiece is not feasible.

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Nomenclature

a, *b*, h – dimension in the *x*-, *y*- and

- z-directions, [m]
- c specific heat, [Jkg⁻¹°C⁻¹]
- I_0 intensity at the center of the spot, [Wm⁻²]
- q_{μ} heat flow absorbed by the workpiece, [J]
- q''_{m} heat flux, $[Jm^{-2}]$
- q''' heat generation, [Jm⁻²] r - laser spot radius, [m]
- r laser spot radius, [m] s – laser spot area, [m²]
- T actual temperature, [°C]
- t = time, [s]

- T_0 surrounding environment temperature, [°C]
- v velocity of the laser beam, [ms⁻¹]
- x, y, z Cartesian co-ordinates, [m]
- ΔT temperature increment, [°C]

Greek symbols

- α thermal diffusivity [m²s⁻¹]
- δ dirac delta function
- λ workpiece thermal conductivity, [Wm⁻¹°C⁻¹]
- ρ density [kgm⁻³]
 - defisity [kgfif *]

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