ANALYSIS OF HEAT AND MASS TRANSFER MECHANISM DURING THERMAL ENERGY STORAGE AND TEMPERATURE REGULATION

by

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Original scientific paper https://doi.org/10.2298/TSCI191116109D

To strengthen the heat and mass transfer capacity and improve the temperature regulation rate, potential storage is taken as the research object in this research to study the heat energy storage of the battery in the low temperature environment. Lattice Boltzmann method is adopted to study the heat energy storage influence mechanism of the temperature regulation system of the low temperature phase-change materials. In addition, the influence of different physical parameters (thermal conductivity and latent heat of phase change) on the thermal insulation of the system in the process of temperature control is revealed. The results show that the mechanism of heat and mass transfer in the process of heat storage and temperature control is related to the different physical properties of phase change materials. The decrease of thermal conductivity and the increase of latent heat of phase change materials will greatly increase the effect of heat energy storage. Therefore, under the action of phase change latent heat, phase change material can effectively extend the holding time of the battery in the low temperature environment.

Key words: potential storage, battery, latent heat of phase change, thermal conductivity

Introduction

The development of human society cannot be separated from the use of energy. With the progress of human beings, the social demand for energy is constantly increasing [1]. The world's energy resources are mainly non-renewable resources (coal and oil). A large amount of carbon, sulfur, and other elements released after combustion are discharged into the environment in the form of CO_2 , sulfide, and other compounds. With the shortage of limited resources, environmental pollution is becoming more and more serious [2, 3]. At this time, wind energy, solar energy, and other clean energy come into being. However, due to the limited energy exploitation technology under natural conditions, the energy utilization is uneven and unstable. Therefore, in order to balance the unreasonable energy supply and energy loss in many energy utilization systems and avoid unnecessary large amount of energy waste, energy storage technology emerges at the historic moment, that is, heat energy storage.

Heat energy storage is divided into two parts, first, the sensible heat storage, second, latent heat (LH) storage. Where, LH storage is to take solid-liquid phase change material (PCM) as the energy storage medium and adopt the material to absorb or release heat during phase change to store energy [4-6]. The LH storage is characterized by large energy storage

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density, flexible design, and easy management. It has a practical development prospect in alleviating the problem of energy shortage [7-9]. In this research, on the basis of LH storage, lattice Boltzmann model is established through temperature regulation of low temperature PCM to explore the heat transfer and mass transfer mechanism of heat storage of the system, providing reference for the improvement of heat storage [10-12].

Methods

In order to study the thermal insulation property of the PCM, a numerical control model of temperature control is established based on lattice Boltzmann method, and fatty acids are selected as PCM [13-15]. The thermophysical properties of batteries and fatty acid PCM are shown in tab. 1.



Table 1. Thermal properties of batteries and fatty acid PCM

Material	λ_0	C_p	ρ	μ	$h_{ m slo}$	β
Battery	4	1000	2600	—	—	_
Fatty acids	0.4	1740	940	9.2 · 10 ⁻⁴	198000	10-4

The reference physical properties are used to assist the analysis, and the differences of physical properties of solid-solid PCM and liquid-liquid PCM are ignored to study the influence of the thermal physical properties of fatty acids. On this basis, a schematic diagram of temperature control of PCM battery at low temperature is established, as shown in fig. 1. The height (AB) of the system is 67 mm. The AB is taken as the axis of symmetry, where the length of AC and EC are equal, both 14 mm. The ECDF area is the PCM, and the ABCD area is the battery. At the beginning of the simulation, the temperature on one side of the PCM drops to the cold wall temperature, at which point the cold wall temperature is less than the temperature of PCM. Other walls remain adiabatic and ignore the contact thermal resistance in this environment [16, 17].

Figure 1. Schematic diagram of temperature control of PCM battery at low temperature

The PCM involved in this research are all regarded as incompressible and Newtonian fluids, and their mass and momentum conservation equations are [18, 19]:

$$\frac{\partial \rho}{\partial t} + \nabla(\rho u) = 0 \tag{1}$$

$$\frac{\partial(\rho u)}{\partial t} + \nabla(\rho u u) = -\nabla\rho + \nabla(\mu \nabla u) + F$$
⁽²⁾

where ρ [kgm⁻³] is the density, u [ms⁻¹] – the velocity, t [s] – the time, μ [Pas⁻¹] – the dynamic viscosity, and F [Nm⁻³] – the volume force applied to the fluid:

$$\rho \frac{\partial H}{\partial t} + \rho \nabla (Hu) = \nabla (\lambda \nabla T)$$
(3)

here H [Jkg⁻¹] is enthalpy.

In this study, Lattice-Boltzmann equation is:

$$f_i(x + e_i\Delta t, t + \Delta t) - f_i(x, t) \equiv \Omega(x, t) + \Delta t F_i$$
(4)

where $e_i \,[\text{ms}^{-1}]$ is discrete velocity, $f_i \,[\text{kgm}^{-3}]$ – the function of density distribution, $F_i \,[\text{kgm}^{-3}\text{s}^{-1}]$ – the discrete term of external force in velocity space, and Ω – the collision term.

$$g_i(x+e_i\Delta t, t+\Delta t) = g_i(x,t) - \frac{1}{\tau_g} \Big[g_i(x,t) - g_i^{\rm eq}(x,t) \Big]$$
(5)

where g_i [Jkg⁻¹] is the distribution function of total enthalpy, and g_i^{eq} [Jkg⁻¹] – the corresponding function of equilibrium distribution.

There is no flow state inside the cell, so only the heat transfer equation needs to be derived. The equation of energy conservation of the battery:

$$\frac{\partial T}{\partial t} = \nabla(\alpha_{\rm b} \nabla T) \tag{6}$$

where the subscript b is the battery. The corresponding lattice Boltzmann evolution:

$$n_{i}(x + e_{i}\Delta t, t + \Delta t) = n_{i}(x, t) - \frac{1}{\tau_{g}} \Big[n_{i}(x, t) - n_{i}^{\text{eq}}(x, t) \Big]$$
(7)

and temperature equilibrium distribution:

$$n_i^{\rm eq} = \omega_i T \tag{8}$$

Figure 1 shows battery temperature control of PCM that at intermediate contact boundary, the battery and PCM need to exchange information, and the battery and PCM need to apply two sets of lattice methods.:

As shown in fig. 2, the control body of the lattice node of acef is taken as an example. In the left half of the control body "adbf" belongs to the battery area. In the right half "dceb" belongs to the PCM region. Taking "db" as interface, energy conservation is constructed in "adbf" region, and the discrete format of eq. (9) can be obtained. Similarly, energy conservation is constructed in the "dceb" region, and the discrete format of eq. (10) can be obtained. The Q is the heat exchange through the interface "db". The following two equations can be combined to obtain the temperature value at the next moment:



Figure 2. Schematic diagram of temperature control of PCM battery at low temperature

$$(\rho C_P)_P \frac{\Delta x^2}{2} \left(T_{i,j}^{t+\Delta t} - T_{i,j}^t \right) = \lambda_b \Delta x \Delta t \left[\frac{T_{i,j}^t - T_{i-\Delta x,j}^t}{\Delta x} + \frac{1}{2} \left(\frac{T_{i,j}^t - T_{i,j+\Delta x}^t}{\Delta x} + \frac{T_{i,j}^t - T_{i,j-\Delta x}^t}{\Delta x} \right) \right] - Q \tag{9}$$

$$(\rho C_P)_{\text{PCM}} \frac{\Delta x^2}{2} \left(T_{i,j}^{t+\Delta t} - T_{i,j}^t \right) = \lambda_{b\text{PCM}} \Delta x \Delta t \left[\frac{T_{i,j}^t - T_{i+\Delta x,j}^t}{\Delta x} + \frac{1}{2} \left(\frac{T_{i,j}^t - T_{i,j+\Delta x}^t}{\Delta x} + \frac{T_{i,j}^t - T_{i,j-\Delta x}^t}{\Delta x} \right) \right] + Q \quad (10)$$

As the battery runs, it releases heat, causing the battery itself and the PCM to warm up. Before the simulation starts, it is assumed that all the PCM have become liquid and the battery has stopped working at the initial time of calculation, so as to explore properties of thermal insulation of PCM [20]. Phase change temperature of PCM on the right wall surface is fixed at T_m , and battery temperature is evenly distributed and maintained at T_0 , where T_0 is greater than T_m . Thus, the convergence benchmark obtained after steady-state calculation of its working condition:

$$\frac{\sum \left|\Gamma^{t+\Delta t} - \Gamma^{t}\right|}{\sum \Gamma^{t}} < 10^{-9} \tag{11}$$

where Γ is temperature and velocity. Then the system variables are dimensionless again. The T^* , t^* , Ra, and ψ , respectively, represent the dimensionless physical quantities used in the study. These physical quantities are all non-quantized processing of physical quantities in the total enthalpy phase change lattice Boltzman model. The dimensionless processing of them is to better study the heat and mass transfer law of phase change materials. Different dimensionless physical quantities are defined:

$$T^* = \frac{T - T_m}{T_0 - T_m}$$
(12)

$$t^* = \frac{\alpha_b t}{L^2} \tag{13}$$

$$Ra = \frac{g\beta(T_0 - T_m)L^3}{v_{PCM}\alpha_{PCM}}$$
(14)

$$Ra = \frac{C_{pPCM}(T_0 - T_m)}{h_{sl}}$$
(15)

$$\psi_{\lambda} = \frac{\lambda}{\lambda_0} \tag{16}$$

$$\psi_{\lambda} = \frac{h_{sl}}{h_{sl0}} \tag{17}$$

The distance AB in fig. 1 is selected as the characteristic length, where the TC of the PCM is between 0.2 and 0.3 W/mK, and the LH of phase change is between 185000 and 205000 J/kg. Initial temperature T_0 of the battery is 10 °C (=1) higher than that of T_m , and the temperature range of dimensionless environment after cooling is between -1.5 °C and -0.5 °C. The expression of the average Nusselt number on the left boundary of the phase change material:

$$Nu_{ave} = \frac{1}{L} \int_{0}^{L} (Nu)_{0} dy = \frac{1}{L} \int_{0}^{L} \left(\frac{\partial}{\partial \partial} \right) dy$$
(18)

The Nusselt number is obtained from eq. (18). The contact surface CD in fig. 1 replaces the left wall surface of the system. The dimensionless average temperature, T_{ave}^* , of phase change materials can be expressed:

$$T_{\text{ave}}^* = \frac{\int T^* ds}{\int \int ds}$$
(19)

The dimensionless temperature standard deviation, σ , of phase change material can be expressed:

$$\sigma = \sqrt{\frac{\int (T^* - T^*_{ave})^2 ds}{\int_s ds}}$$
(20)

where $S \text{ [m^2]}$ is the battery area, and the mean temperature and temperature standard deviation of the battery are obtained from eqs. (19) and (20). The solution region is ABEF in fig. 1. In this research, the grid system is selected as 52×130 , 78×195 , 104×260 , and 130×325 ($\psi_{\lambda} = 1$, $\psi_h = 1$, $T_c^* = -1$) for solution, and the grid Boltzmann model is tested for independent solution. The results show that when the grid system is encrypted from $104 \times 260-130 \times 325$, the average relative error of Nusselt number between them is less than 1%. Therefore, the grid system used in this research is 104×260 .

Results

Thermal conductivity

The effect of the temperature change (TC) of PCM on thermal insulation performance of battery temperature control system is as follows.

Figure 3 shows the curve of average temperature change of battery. The ψ_{λ} is between 0.66 and 1. In the initial process, due to the fact that most of the PCM are liquid phase, the temperature of the battery decreases rapidly based on the influence of natural-convection. When the PCM begins a liquid-solid phase change, the LH stored inside is released, and temperature drop rate of the battery gradually slows down. Different from heat dissipation, the battery temperature control system in a low temperature environment needs to maintain the temperature. Therefore, increasing the TC of the PCM will accelerate heat dissipation of battery to the environment.



Figure 3. Battery average temperature curve

Figure 4 shows the average Nusselt number of CD surface under different TC of PCM. Average Nusselt number is the dimensionless temperature gradient on the CD surface. It can be observed that the main source of heat transfer on the CD surface in the initial stage is convection. When the heat diffuses to the environment, PCM gradually transforms from the liquid phase to the solid phase, and phase interface gradually moves to the direction of the battery. The natural-convection area begins to decrease, so the heat transfer of interface decreases, and the average Nusselt number decreases. When PCM is completely transformed into solid phase,



Figure 4. The CD surface average Nusselt number of PCM with different thermal conductivity

heat transfer on wall increases rapidly, accelerating the increase of the average Nusselt number. The average Nusselt number gradually flattens, when TC reduces.

Figure 5 shows the change curve of total liquid phase ratio under different TC of PCM. When TC increases gradually, the decline rate of liquid phase rate increases gradually.



Figure 5. Curve of total liquid phase rate under TC of different PCM

The LH of phase change

Figure 6 shows the variation curve of the average battery temperature with ψ_h . As mentioned previously, the average temperature of the battery decreases gradually before the PCM is completely converted to solid phase. It can be concluded that, at the same time, the average temperature difference of the battery under different phase change LH is small. However, the temperature of the battery drops rapidly after PCM is completely converted to solid. Therefore, solidification time of PCM can be effectively extended by large LH of phase change.

Figure 7 represents change curve of the average Nusselt number on the CD surface. After the PCM is completely solidified, the average Nusselt number at the contact surface increases rapidly without the effect of LH of phase change, and heat transfer increases, so that temperature of the battery drops rapidly. Before the PCM solidifies completely, the LH of PCM could be increased to increase heat transfer at the contact surface.

Figure 8 shows the curve of liquid phase rate under LH of different PCM. It is found that increasing LH of PCM can improve thermal insulation performance of PCM. As mentioned previously, reducing the TC of PCM and weakening heat transfer property of the PCM can im-

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prove the insulation capacity of the system. However, considering that the temperature control system needs to heat the battery at high temperature, if TC is insufficient, heat transfer property of temperature control system will be limited, so that system cannot have both heat dissipation and heat preservation functions. What's more, the LH of phase change can be increased to maintain the working temperature of the battery for a long time when the temperature control system dissipates heat. The change of TC is usually the addition of low TC materials for heat

insulation, so that the mass of PCM per unit volume decreases, and LH decreases relatively. Therefore, the primary task of temperature control system of low temperature PCM is to ensure LH of phase change.

Discussion

Heat storage and heat transfer mechanism are discussed, and the constant temperature characteristics of fatty acid PCM are analyzed. The thermal insulation performance of the temperature control system of PCM under low temperature environment is studied, and the influence rule of different physical parameters and environmental temperature on the temperature control performance is revealed. In the temperature control process of low temperature of PCM, under the action of the LH of the phase change, the holding time of the battery in the low temperature environment can be effectively extended by adopting PCM, and a small temperature standard deviation of battery can be maintained.

As mentioned previously, the TC of PCM is generally low. According to Fourier's heat conduction rule, the value of heat transfer rate is correlated with the thermal conductivity. The lower the TC is, the slower the propagation rate will be, which will reduce the storage rate and heat dissipation rate of heat energy storage and increase the time required for heat energy storage. Therefore, the basic heat and mass transfer properties of PCM can be enhanced and the rate of LH storage can be improved by utilizing heat storage and mass transfer mechanism in the process of temperature control. Under the current demand of high energy storage density, only a single heat transfer form cannot meet the needs of the technology. In the future, multiple enhancement methods should be developed for common heat transfer enhancement. In view of the low TC of some PCM, carbon materials such as high TC metal particles, expanded graphite, and graphene can be added, and high TC fins can be inserted, or the heat transfer structure can be designed, so as to provide reference value for the heat storage process.

Conclusion

One of the most effective methods for heat and mass storage is phase change energy storage. Heat energy storage can improve energy efficiency and solve the problem of uneven energy distribution in time and space. Because of its stable working temperature and high density storage characteristics, it has become the research object of experts at home and abroad. In this research, the thermal insulation performance of the battery in a low temperature environment is studied, and with the help of lattice Boltzmann method, the characteristics of temperature control of PCM are analyzed, which reveals the specific heat and mass transfer mechanism of the melting rate and temperature distribution of the PCM. Firstly, under the action of phase change LH, the use of PCM can effectively extend the holding time of battery at low temperature. During the solidification of PCM, the temperature standard deviation of the battery gradually decreases because of LH of the phase change, which limits temperature change of battery. When the PCM solidifies completely, the temperature standard deviation of the battery increases rapidly. Secondly, decreasing the TC of the PCM and increasing LH of PCM will prolong the holding time. Changing LH of the PCM in initial solidification stage has little influence on temperature magnitude and distribution of battery, but it will change the complete solidification time of the PCM. Considering that the PCM temperature control system needs to dissipate the heat of the battery under high temperature environment, the PCM with high LH can better meet the requirements of thermal insulation. Therefore, the decrease of TC and increase of phase change LH are beneficial to heat energy storage and enhancement of heat and mass transfer in process of temperature control, so as to improve the rate of LH storage.

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Acknowledgment

National Natural Science Foundation of China (No. 51705332).

References

- Dong, D., Thermal Optimisation of Metal Hydride Reactors for Thermal Energy Storage Applications, Sustainable Energy and Fuels, 1 (2017), 8, pp. 1820-1829
- [2] Xu, A., et al., Lattice Boltzmann Modeling of Transport Phenomena in Fuel Cells and Flow Batteries, Acta Mechanica Sinica (English Series), 33 (2017), 3, pp. 555-574
- [3] Wang, W., The Fe₃O₄ Functionalized Graphene Nanosheet Embedded Phase Change Material Composites: Efficient Magnetic and Sunlight-Driven Energy Conversion and Storage, *Journal of Materials Chemistry*, *A5* (2017), 3, pp. 958-968
- [4] Deng, Y., Preparation and Characterization of KNO₃-Diatomite Shape-Stabilized Composite Phase Change Material for High Temperature Thermal Energy Storage, *Journal of Materials Science and Tech*nology, 33 (2017), 2, pp. 198-203
- [5] Ye, W.-B., Enhanced Latent Heat Thermal Energy Storage in the Double Tubes Using Fins, *Journal of Thermal Analysis and Calorimetry*, 128 (2017), 1, pp. 533-540
- [6] Kandasamy, H., et al., Investigation on Phase Change Behavior of Paraffin Phase Change Material in a Spherical Capsule for Solar Thermal Storage Units, *Heat Transfer Engineering*, 39 (2018), 9, pp. 775-783
- [7] Abdulateef A. M., Thermal Performance Enhancement of Triplex Tube Latent Thermal Storage Using Fins-Nanophase Change Material Technique, *Heat Transfer Engineering*, 39 (2018), 12, pp. 1067-1080
- [8] Qi, G., et al., Hierarchical Graphene Foam-Based Phase Change Materials with Enhanced Thermal Conductivity and Shape Stability for Efficient Solar-to-Thermal Energy Conversion and Ctorage, Nano Research, 10 (2017), 3, pp.802-813
- [9] Wen, R., Thermal Energy Storage Properties and Thermal Reliability of PEG-Bone Char Composite as a Form-Stable Phase Change Material, *Journal of Thermal Analysis and Calorimetry*, 132 (2018), 3, pp. 1753-1761
- [10] Mahdi, J. M., Nsofor, E. C., Solidification of a PCM with Nanoparticles in Triplex-tube Thermal Energy Storage System, *Applied Thermal Engineering*, 108 (2016), Sept., pp. 596-604
- [11] Mahdi, J. M., Nsofor, E. C., Melting Enhancement in Triplex-tube Latent Heat Energy Storage System Using Nanoparticles-metal Foam Combination, *Applied Energy*, 191 (2017), Apr., pp. 22-34
- [12] Qian, T., et al., Adjustable Thermal Property of Polyethylene Glycol/Diatomite Shape-Stabilized Composite Phase Change Material, Polymer Composites, 37 (2016), 3, pp. 854-860
- [13] Mahdi, J. M., Nsofor, E. C., Solidification Enhancement of PCM in a Triplex-tube Thermal Energy Storage System with Nanoparticles and Fins, *Applied Energy*, 211 (2018), Feb., pp. 975-986
- [14] Espinoza, M., et al., Lattice Boltzmann Modeling From the Macro- to the Microscale An Approximation to the Porous Media in Fuel Cells, Proceedings, REGWA Symposium 2014 Regenerative Energy and Hydrogen Technology Symposium, Stralsund, Germany, 2014
- [15] Weckerle, C., et al., Numerical Optimization of a Plate Reactor for a Metal Hydride Open Cooling System, International Journal of Hydrogen Energy, 44 (2019), 31, pp. 16862-16876
- [16] Fu, X., et al., Preparation and Properties of Lauric Acid/Diatomite Composites as Novel Form-Stable Phase Change Materials for Thermal Energy Storage, Energy and Buildings, 104 (2015), Oct., pp. 244-249
- [17] Jayakumar, A., et al., A Technical Review on Gas Diffusion, Mechanism and Medium of PEM Fuel Cell, 21 (2015), 1, pp. 1-18
- [18] Mahdi, J. M., Nsofor, E. C., Solidification Enhancement in Triplex-Tube Latent Thermal Energy Storage System Using a Combination of Nanoparticles and Fins, *Proceedings*, ASTFE 2th Thermal and Fluids Engineering Conf., Las Vegas, Nev., USA, 2017
- [19] Jung, D. I., et al., An Experimental Study on the Heat Transfer Characteristics of a Finned-Tube Heat Exchanger in a PCM Thermal Energy Storage System, Korean Journal of Air Conditioning & Refrigeration Engineering, 28 (2016), Jan., pp. 15-20
- [20] Chen, L., et al., Experimental Investigation of Plastic Finned-Tube Heat Exchangers with Emphasis on Material Thermal Conductivity, Experimental Thermal & Fluid Science, 33 (2009), 5, pp. 922-928

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