

CRIMP FREQUENCY OF A VISCOELASTIC FIBER IN A CRIMPING PROCESS

by

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Crimp frequency during the stuffer box crimping process is of great importance for controlling morphology of crimped fibers. A fiber is considered as a viscoelastic rheological fluid, and an approximate formulation for crimp frequency is obtained revealing the main factors affecting the crimping process.

Key words: stuffer box crimping, crimped fiber, viscoelastic rheological model, crimped frequency

Introduction

Stuffer box crimping [1-4] is widely used for fabrication of microscale crimped fibers, and its mechanism is also used for fabrication of nanoscale crimped fibers [4] by bubble electrospinning [5, 6] or bubbfil spinning [7-10]. The governing equations were established for the transverse vibration of an axially moving fluid arising in the stuffer box crimping by the Hamilton's principle [1]. The viscoelastic fluid is subject to a complex vibration, and it will be crimped after solvent evaporation. The crimp frequency will greatly affect the morphology and properties of the crimped fibers. This paper search for an approximate frequency formulation to elucidate the main factors affecting the crimping process, so that the process can be optimized and controlled.

Governing equation

Governing differential equations for an axially moving slender fiber of viscoelastic fluid were obtained in [1]. The previous study revealed that the velocity of the axial motion affects greatly the natural frequencies of transverse vibration. If transverse vibration occurs just before fiber solidification, a crimped fiber can be obtained, and the crimped frequency can be adjusted by the velocity of axial motion [1]. This paper focuses on the effect of fiber viscoelastic rheological property on the crimping process. The viscoelastic rheological model for a moving fiber with axial stiffness EA and flexural rigidity EI is adopted in the form:

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$$\sigma_L = E\varepsilon_L + \eta \frac{D\varepsilon_L}{Dt} = E\varepsilon_L + \eta \left(\frac{\partial \varepsilon_L}{\partial t} + u \frac{\partial \varepsilon_L}{\partial x} \right) \quad (1)$$

where σ_L is the axial stress, η – the parameter for viscoelastic rheological property, ε_L – the non-linear strain, D/Dt – the material derivative of the transverse displacement, defined:

$$\frac{D}{Dt} = \frac{\partial}{\partial t} + u \frac{\partial}{\partial x} \quad (2)$$

where u is the velocity of the moving fiber. The non-linear strain is defined:

$$\varepsilon_L = \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 \quad (3)$$

where w is the transverse displacement.

The governing equation was obtained by Hamilton principle [1], which is:

$$\begin{aligned} & \rho A \frac{\partial^2 w}{\partial t^2} + 2u\rho A \frac{\partial^2 w}{\partial x \partial t} + \rho A u^2 \frac{\partial^2 w}{\partial x^2} - N \frac{\partial^2 w}{\partial x^2} - \\ & - \frac{3}{2} AE \frac{\partial}{\partial x} \left(\frac{\partial w}{\partial x} \right)^2 \frac{\partial^2 w}{\partial x^2} - \eta A \left(\frac{dw}{dx} \right) \frac{\partial^2 w}{\partial t \partial x} \frac{\partial^2 w}{\partial x^2} - \frac{1}{2} \eta A \left(\frac{dw}{dx} \right)^2 \frac{\partial^3 w}{\partial t \partial x^2} - \\ & - \eta A u \left(\frac{\partial w}{\partial x} \right) \left(\frac{\partial^2 w}{\partial x^2} \right)^2 - \frac{1}{2} \eta A u \left(\frac{\partial w}{\partial x} \right)^2 \left(\frac{\partial^3 w}{\partial x^3} \right) + EI \frac{\partial^4 w}{\partial x^4} = 0 \end{aligned} \quad (4)$$

where w is the transverse displacement, A – the section area, u – the velocity of the moving fiber, F – the liquid fiber tension per area, P – the fluid pressure, and ρ – the density.

Crimp frequency

We assume that the solution of eq. (4) can be expressed in the form:

$$w = w_0 + \eta w_1 \quad (5)$$

where w_0 and w_1 are solved, respectively, from the following equations:

$$\rho A \frac{\partial^2 w_0}{\partial t^2} + 2u\rho A \frac{\partial^2 w_0}{\partial x \partial t} + \rho A u^2 \frac{\partial^2 w_0}{\partial x^2} - N \frac{\partial^2 w_0}{\partial x^2} - 3AE \frac{\partial w_0}{\partial x} \left(\frac{\partial^2 w_0}{\partial x^2} \right)^2 + EI \frac{\partial^4 w_0}{\partial x^4} = 0 \quad (6)$$

and

$$\begin{aligned} & \rho A \frac{\partial^2 w_1}{\partial t^2} + 2u\rho A \frac{\partial^2 w_1}{\partial x \partial t} + \rho A u^2 \frac{\partial^2 w_1}{\partial x^2} - N \frac{\partial^2 w_1}{\partial x^2} - \\ & - \frac{3}{2} AE \frac{\partial}{\partial x} \left(\frac{\partial w_0}{\partial x} \right)^2 \frac{\partial^2 w_1}{\partial x^2} - \frac{3}{2} AE \frac{\partial}{\partial x} \left(\frac{\partial w_1}{\partial x} \right)^2 \frac{\partial^2 w_0}{\partial x^2} - A \left(\frac{dw_0}{dx} \right) \frac{\partial^2 w_0}{\partial t \partial x} \frac{\partial^2 w_0}{\partial x^2} - \frac{1}{2} A \left(\frac{dw_0}{dx} \right)^2 \frac{\partial^3 w_0}{\partial t \partial x^2} - \\ & - Au \left(\frac{\partial w_0}{\partial x} \right) \left(\frac{\partial^2 w_0}{\partial x^2} \right)^2 - \frac{1}{2} Au \left(\frac{\partial w_0}{\partial x} \right)^2 \left(\frac{\partial^3 w_0}{\partial x^3} \right) + EI \frac{\partial^4 w_1}{\partial x^4} = 0 \end{aligned} \quad (7)$$

The solution of eq. (6) can be presented in the form:

$$w_0(x, t) = W_0(x) \cos \omega t \quad (8)$$

where W_0 is the normal function, ω – the natural frequency, and $N = FA - PA$.

The differential equation for the mode shape of vibration, after submitting eq. (8) into eq. (6), is obtained:

$$-\omega^2 \rho A W_0 + (\rho A u^2 - N) W_0'' + EI W_0^{(4)} = 0 \quad (9)$$

The expression of the normal function is assumed to have the form:

$$W_0 = w_{\max} \sin\left(\frac{\pi}{L} x\right) \quad (10)$$

we obtain the following fundamental frequency of vibration:

$$\omega = \sqrt{-\frac{\pi^2}{\rho A L^2} (\rho A u^2 - N) + \frac{\pi^4}{\rho A L^4} EI} \quad (11)$$

The normal mode shape of vibration reads:

$$w_0(x, t) = w_{\max} \sin\left(\frac{\pi}{L} x\right) \cos \omega t \quad (12)$$

Now submitting eq. (12) into eq. (7) results in a differential equation for $w_1(x, t)$:

$$\begin{aligned} & \rho A \frac{\partial^2 w_1}{\partial t^2} + 2u \rho A \frac{\partial^2 w_1}{\partial x \partial t} + \rho A u^2 \frac{\partial^2 w_1}{\partial x^2} - N \frac{\partial^2 w_1}{\partial x^2} + EI \frac{\partial^4 w_1}{\partial x^4} - \\ & - \frac{3}{2} AE \frac{\partial}{\partial x} \left(\frac{\partial w_0}{\partial x} \right)^2 \frac{\partial^2 w_1}{\partial x^2} - \frac{3}{2} AE \frac{\partial}{\partial x} \left(\frac{\partial w_1}{\partial x} \right)^2 \frac{\partial^2 w_0}{\partial x^2} - A \left(\frac{dw_0}{dx} \right) \frac{\partial^2 w_0}{\partial t \partial x} \frac{\partial^2 w_0}{\partial x^2} - \frac{1}{2} A \left(\frac{dw_0}{dx} \right)^2 \frac{\partial^3 w_0}{\partial t \partial^2 x} - \\ & - Au \left(\frac{\partial w_0}{\partial x} \right) \left(\frac{\partial^2 w_0}{\partial^2 x} \right)^2 - \frac{1}{2} Au \left(\frac{\partial w_0}{\partial x} \right)^2 \left(\frac{\partial^3 w_0}{\partial^3 x} \right) = 0 \end{aligned} \quad (13)$$

We assume that the second mode shape of vibration can be expressed in the form:

$$w_1(x, t) = \varepsilon \sin\left(\frac{\pi}{2L} x\right) \cos \omega t \quad (14)$$

Submitting eq. (14) into eq. (13) yields a residual equation, setting the residual equation be zero at $x = L$, $t = 0$, we obtain:

$$\varepsilon \omega^2 \rho A + \frac{\pi^2 (\rho A u^2 - N)}{4L^2} + \frac{\pi^4 EI}{16L^4} = 0 \quad (15)$$

From eq. (14), ε can be identified, which is:

$$\varepsilon = -\frac{1}{\omega^2 \rho A} \left[\frac{\pi^2 (\rho A u^2 - N)}{4L^2} + \frac{\pi^4 EI}{16L^4} \right] \quad (16)$$

Therefore, we obtain the mode shape of vibration of the fiber:

$$w(x, t) = w_{\max} \sin\left(\frac{\pi}{L} x\right) \cos \omega t + \varepsilon \eta \sin\left(\frac{\pi}{2L} x\right) \cos \omega t \quad (17)$$

where ω is defined in eq. (11), and ε is defined in eq. (16).

Discussion and conclusion

The transverse vibration of an axially moving slender fiber of viscoelastic fluid can be approximately expressed by eq. (17) for normal mode shape of vibration and second mode shape of vibration. The viscoelastic property, η , mainly affects the second mode shape of vibration, and crimp frequency, ω , mainly depends upon the axially moving speed, fiber mechanical property, and geometrical property of the stuffer box. The obtained approximate crimp frequency given in eq. (11) can be used for optimization of the crimping process for practical applications.

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Nomenclature

A – section area
 EA – fiber axial stiffness
 EI – fiber flexural rigidity
 F – liquid fiber tension per area
 L – effective crimp length
 $N = FA - PA$
 P – fluid pressure
 u – velocity of the moving fiber
 w – fiber transverse displacement

w_{\max} – maximal transverse displacement

Greek symbols

ε – small transverse displacement defined in eq. (16)
 ε_L – non-linear strain
 η – viscoelastic parameter
 ρ – the density
 σ_L – axial stress
 ω – fundamental frequency defined in eq. (11)

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